

### **3-BLADE CASSAVA STEM CUTTING MACHINE**

#### **Abstract**

The three blades cassava stem cutter was designed and fabricated in the Federal University of Technology, Owerri with the aim of reducing time expended and drudgery involved in cutting cassava stem for asexual propagation of the root plant. The developed machine has a dimension of 100cm×73.5cm×70cm with three cutting circular blades rotating in an anti-clockwise motion against the cassava stem bundle pushed towards it by a human operator. The machine was evaluated at varying speeds of 515rpm, 578rpm, 610rpm and 652 rpm with cassava bundles holding 5, 10, 15, and 20 stems in a bundle. The machine's cutting efficiency was rated 100% as the stem cutter efficiently cut a cassava bundle of 20 stems at 15seconds seconds at 515rpm and cuts a bundle of 20 stem at 652rpm within 13seconds. The stem cutter was further analyzed for maximum capacity per hour and the result obtained was 19,000stems this output will conveniently satisfy IITA recommendations of plant population of 12,500stands/hectare at a spacing 1m×0.8m.

*Keywords: Cassava, Cuttings, Blades, efficiency, stem and bundle*

#### **1.0 Introduction**

Cutting is a separation technique where a solid body is mechanically divided along a predetermined part using a cutting tool [5]. Cutting is one of the methods used in asexual propagation of root plants [6]. Cassava roots are the second most important staple crop after maize and are propagated by stem cuttings [2]. As a perennial crop, they can grow up to 24m in height and a benchmark for food security for the poor [12] [11]. Cassava stem handling and propagation involve several steps; long cassava stems should be obtained from 10-12 months old up to 2-3 days (not more than 2 weeks) to enable fast sprouting, when ready to plant the stem should be cut to 25 cm length with at least 5-7 nodes on it using sharp tools like a cutlass, secateurs etc. and should be done avoiding bruises. Considering spacing requirements of 1m by 0.8m on the crest of mounds or ridges; cuttings should be planted horizontally with stem completely buried 5cm deep in dry regions and vertically (slanted or angled) with one-third of stem above the soil in wet regions [9].

Nigeria is currently the largest producer of cassava in the world with an estimated capacity of 57 million metric tonnes per year (FAO, 2017) and losses around ₦1 trillion because of poor value addition. Thailand on the other hand is the largest exporter of dried cassava, responsible for about 80 per cent of global trade in the crop and its by-products [7]. Transforming cassava into various by-products in Nigeria has the potentials of improving its food security situation, diversify its manufacturing base, trade balance and improve livelihoods through job creation [3]. In most of the country, cassava cultivation is done manually especially stem cutting operation. To maximize the opportunities present in cassava production, Nigeria must improve or maintain its current world producer status of the crop. This necessitated research on the development of a small-scale cassava stem cutter in the Federal University of Technology, Owerri that will

supplement other efforts made to reduce drudgery, improve handling, affordable and increase cassava production by shortening the time spent during planting.

### *Operation of Machine*

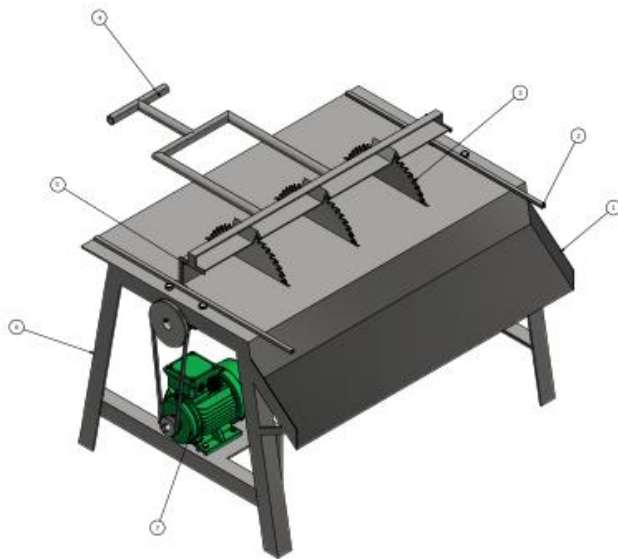


TABLE	
ITEM	DESCRIPTION
1	DISCHARGE CHUTE
2	SLIDER
3	CUTTING BLADE
4	PUSHER HANDLE
5	PUSHER
6	FRAME
7	ELECTRIC MOTOR

*Figure 1. Isometric of a 3-blade cassava stem cutting machine*

The operator positions a stem bundle on top of the machine, parallel to the horizontal pusher (5). Then he pushes the pusher to convey the stems of approximately 100cm of length towards the rotating cutting blades (3) powered by a 2 hp electric motor(7), which cuts and meters the stems into approximate lengths and the cuttings are collected from the discharge chute. The horizontal pusher has a pair of sliders that are lubricated to ease movement. The operator at intervals controls the pusher to reduce stem bending during a cutting operation.

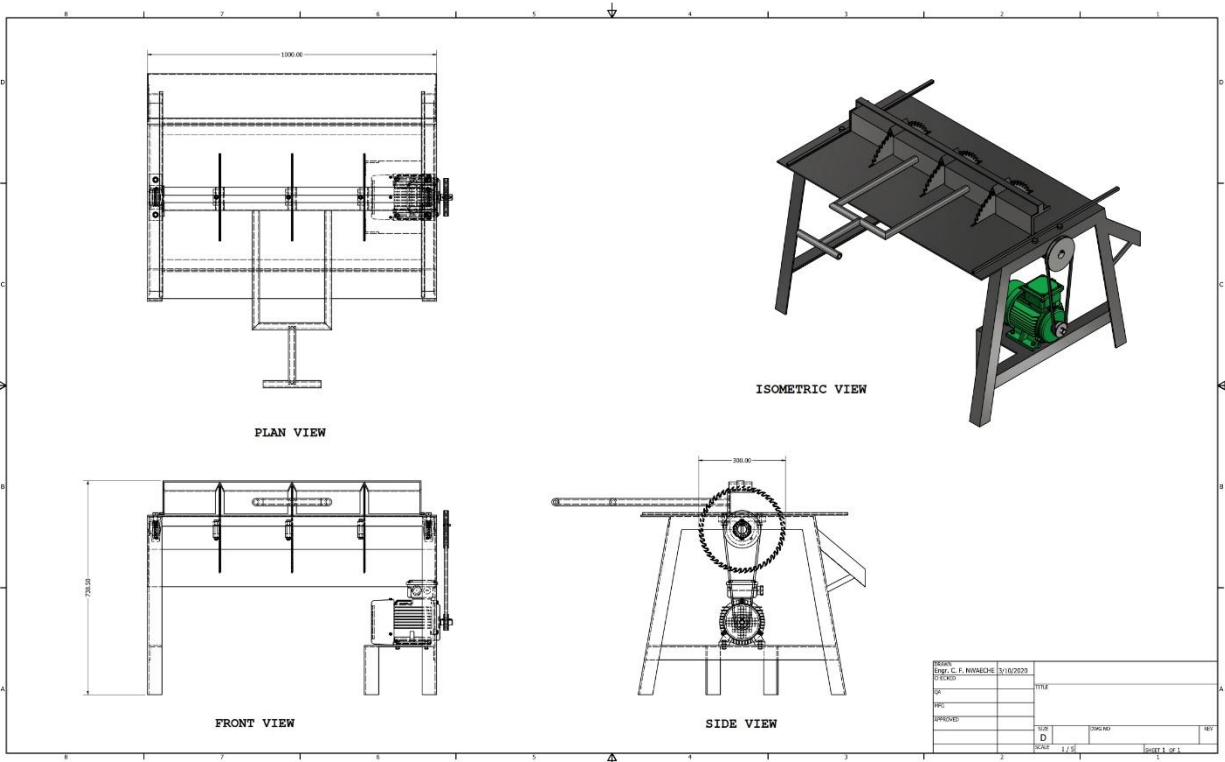


Figure 2. Orthographic view of the 3-blade cassava cutting machine

## 2.0 Design and calculation

The electric motor rating employed is a 2 hp with a speed of 1420rpm.

### Pulley Selection

$$\frac{N_1}{N_2} = \frac{d_1}{d_2}$$

Where  $N_1$  = Speed of electric motor pulley

$N_2$  = Speed of shaft pulley

$d_1$  = Diameter of electric motor pulley used in the design

$d_2$  = Diameter of shaft pulley used in the design

***Length of Belt***

$$L = \pi(r_2 + r_1) + 2x + \frac{(r_2 - r_1)^2}{x}$$

Where  $r_1$  = Electric motor pulley radius

$r_2$  = Shaft pulley radius

$x$  = Centre distance between electric motor and shaft pulley

∴ A type v-belt was selected

***Determination of Belt Angle of Contact and Power Transmitted***

$$\sin\alpha = \frac{r_2 - r_1}{x}$$

$$\theta_1 = 180^\circ - 2\alpha$$

$$\theta_2 = 180^\circ + 2\alpha$$

Where  $\theta_1$  = Angle of contact on the electric motor pulley

$\theta_2$  = Angle of contact on the shaft pulley

∴ Designing for pulley which  $\mu \cdot \theta$  is small

Where  $\mu$  = Coefficient of friction between belt and pulley rim

Then, Belt velocity ( $v$ ),

$$v = \frac{\pi d_1 N_1}{60}$$

*Mass of the belt per meter length = Area × Length × Density*

Centrifugal tension ( $T_c$ ),

$$T_c = m \cdot v^2$$

Maximum tension in the belt ( $T_m$ ),

$$T_m = \text{Allowable tension on belt} \times \text{Area}$$

Total tension in the belt

If  $T_1$  = Tension in the taut side of the belt

$T_2$  = Tension in the slack side of the belt

$$T_m = T_1 + T_c \dots\dots\dots (1)$$

Making  $T_1$  subject formulation in equation (1)

$$T_1 = T_m + T_c$$

$$2.3 \log \left( \frac{T_1}{T_2} \right) = \mu \cdot \theta_2$$

∴ Power transmitted per belt

$$P = (T_1 - T_2)v$$

### ***Driven Shaft Diameter Determination***

Torque transmitted by driven pulley shaft ( $T$ ),

$$T = \frac{\text{Electric motor power} \times 60}{2\pi \times \text{Speed of shaft pulley}}$$

Bending moment on the driven shaft ( $M$ ),

$$M = (T_1 + T_2 + 2T_c)K \times I$$

Where  $K$  = Distance from the centre of the driven pulley to the nearest bearing

$I$  = Number of the belt used

Twisting Moment ( $T_e$ ),

$$T_e = \sqrt{T^2 + M^2}$$

$$\therefore T_e = \frac{\pi}{16} \times \tau \times D^3$$

Where  $\tau$  = Permissible shear stress on the driven shaft

$D$  = Driven Shaft diameter

### ***Force applied and circular blade cutting force***

Actual Force applied by the operator during operation and the cutting force is difficult to determine due to difference in human mass and displacement on the cutting blade geometry during rotation. To identify this displacement on the cutting blades geometry forces applied to each circular blade individual tooth must be put into consideration to determine the total cutting force required [8].

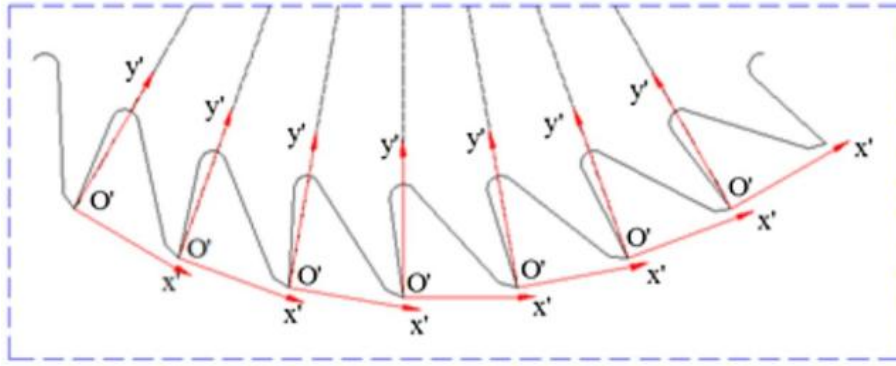


Figure 3. Schematic showing forces acting on individual circular blade tooth. Source: Yazdan Kordestany and Yongsheng Ma

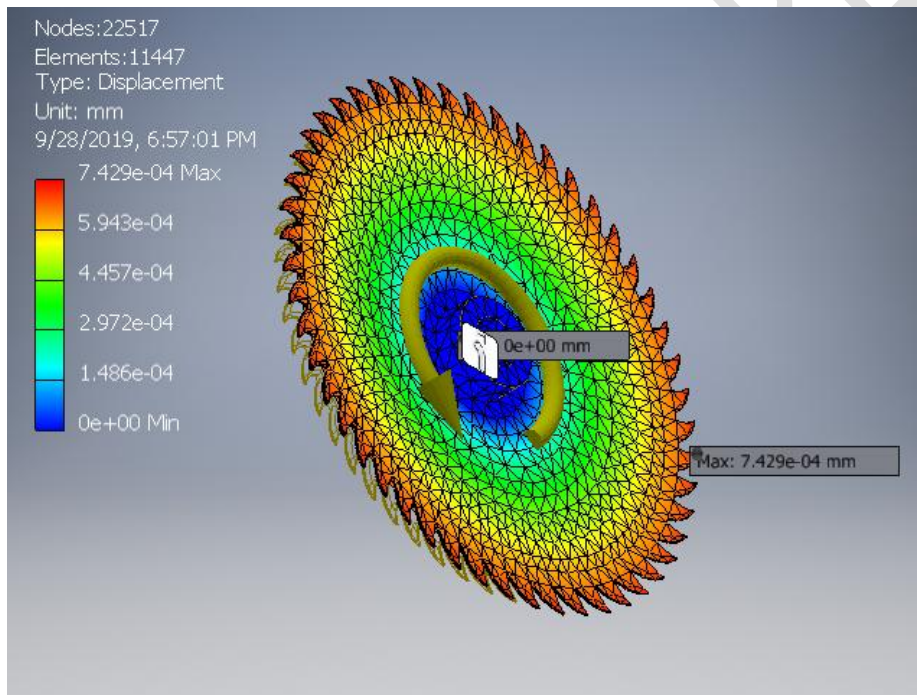


Figure 4. Finite Element Analysis showing displacement across each circular blade

Figure (4) above shows the displacement across the rotating blade as it rotates at the highest rotating speed of 1420rpm. These displacements are represented by different colours. Red colour represents the highest displacement while blue colour represents the lowest displacement.

### ***Predicting Force Applied and work done by Operator***

The average human adult weight in Nigeria is 60.745kg [10].

$$F_F = \mu_K F_N$$

Where  $F_F$  = Frictional force

$\mu_K$  = Coefficient of friction between pusher slider and machine top surface (Metal to Metal

Contact lubricated = 0.03)

$F_N$  = Normal force

$F_N$  = Mass  $\times$  Acceleration due to gravity

$$\begin{aligned} \therefore F_F &= 0.03 \times 60.745\text{kg} \times 9.8\text{m/s}^2 \\ &= 17.859\text{N} \end{aligned}$$

*Work – done (W) =  $F_S \cos \theta$*

Where S = displacement between the pusher and when cassava stakes meet circular blade = 0.15m Approx.

$\theta$  = Angle between F and s = 0

$$\therefore \text{Cos } \theta = 1$$

$$= 17.859 \times 0.15 \times 1 = 2.678 \text{ Joules}$$

### 3.0 Results and discussion

To assess the machine, cutting shaft speeds of 515, 578, 610 and 652 rpm were selected and different stem bundles of 5, 10, 15 and 20 were fed at different time intervals which were done using a stopwatch. The formulas below were used to determine the machine's efficiency and capacity;

$$\text{Cutting efficiency} = \frac{\text{Actual number of cuttings gotten}}{\text{Expected number cuttings from each bundle}} \times 100\%$$

$$\text{Machine output capacity} = \frac{\text{Quantity of cassava cuttings produced}}{\text{time for cutting to be completed (s)}}$$

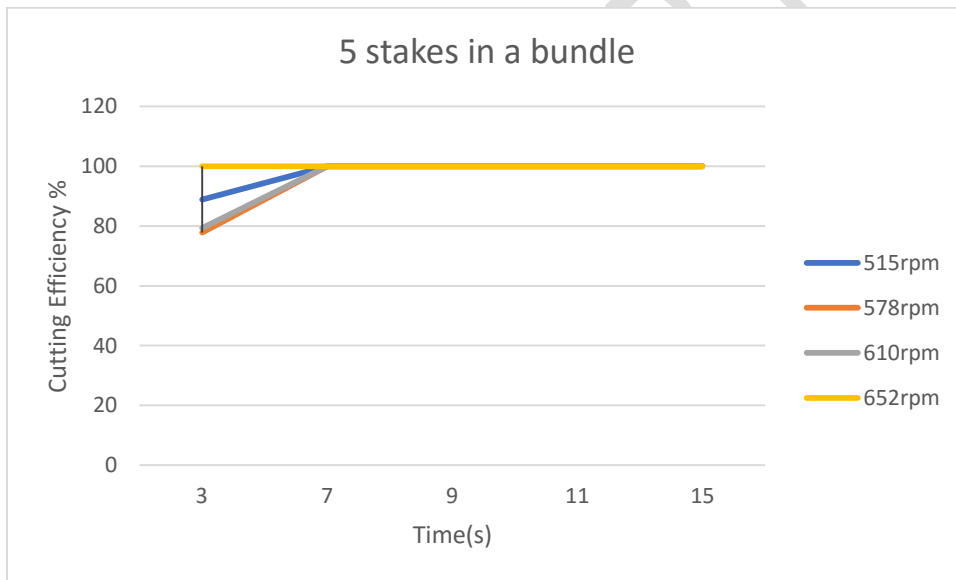


Figure 5. Effect of time on efficiency for various cutting speeds



Figure 6. Effect of time on efficiency for various cutting speeds

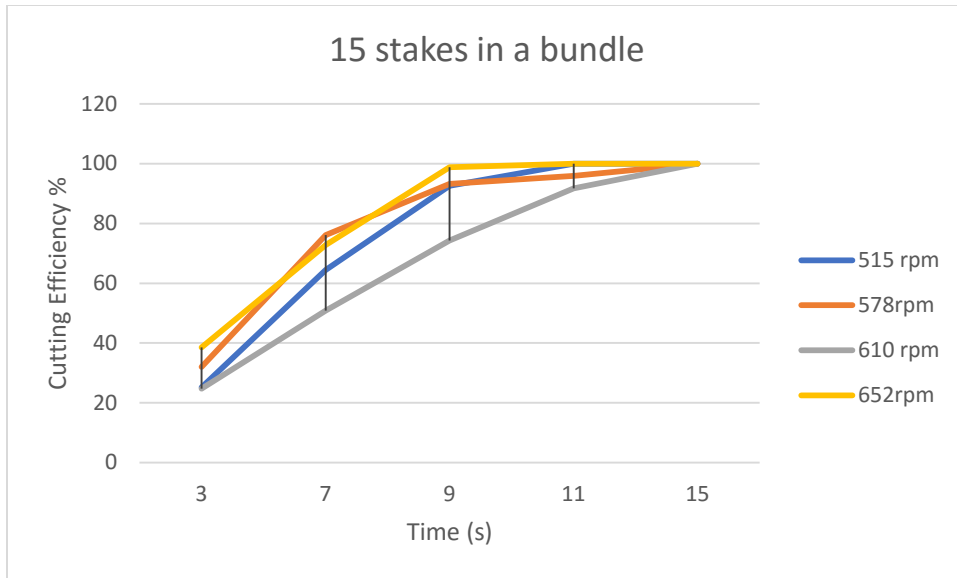


Figure 7. Effect of time on efficiency for various cutting speeds

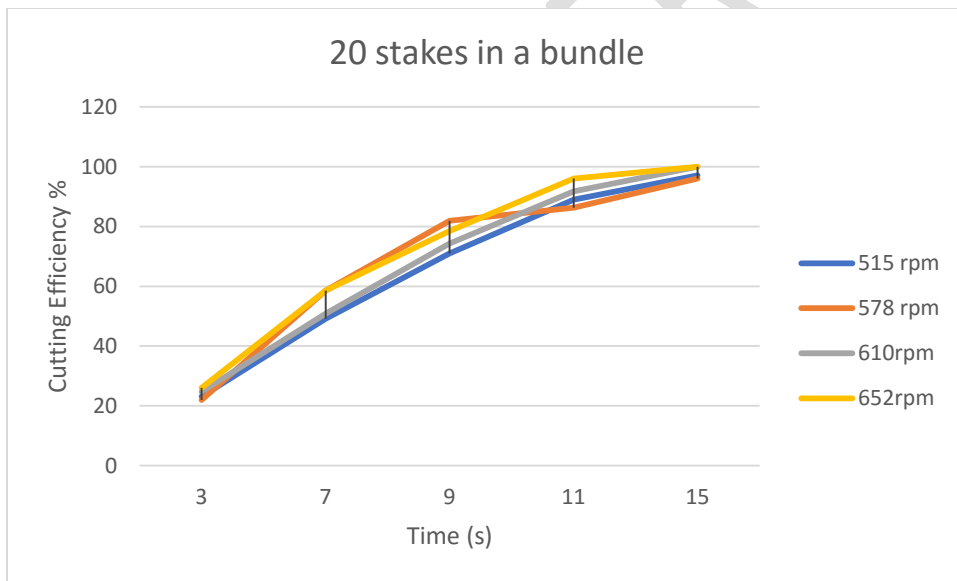


Figure 8. Effect of time on efficiency for various cutting speeds

Figure 5 shows that at 515 rpm speed, the cutting efficiency starts above 80% with 3 seconds into cutting, then gradually increases close to 100% in the space of 3.5 – 4 seconds. The

experience is similar for speeds of 578 and 610 rpm. Under the 652rpm speed, it can be observed that the cutting efficiency starts at 100% and maintains the efficiency throughout

Figure 6 shows that at 515 rpm speed, the cutting efficiency starts above 40 % with 3 seconds into cutting and increases close to 100% in the space of 4 seconds, and then eventually reaches 100% in about 4 seconds later. Almost the same experience is observed for speeds of 578 and 610 rpm. Under the 652 rpm, the experience is again similar except that cutting efficiency starts at 60% under 3 seconds and reaches 100% 4 seconds later while maintaining this efficiency throughout. We can see that an increase in the stakes in a bundle decreases the starting cutting efficiency of the machine while taking a long time to achieve 100% cutting efficiency.

Figures 5, 6, 7 and 8 above shows that there was a significant increase in cutting efficiency as the cutting blade speeds were increased.

Also, the figures show that an increase in residence time had a significant impact on the machines cutting efficiency across all the bundles fed into the machine. 20 stakes with variable stem diameters in a bundle were the maximum capacity the machine can take per batch and it gave the highest cuttings of 19,000/hour which is enough to meet IITA's recommended planting spacing requirement of 1 m × 0.8 m which will give a plant population of 12,500 stands/hectare.

#### **4.0 Conclusion**

The machine's cutting efficiency was 100% meaning it gave the desired number of cuttings at its maximum capacity. This is very satisfactory, so we recommend the machine for medium scale production of cassava stakes to promote cassava production.

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