

Original Research Article
Mechanics of Sport Climbing Falls

Abstract

Rock climbing falls can be safely stopped and are often a routine part of the sport. However, safety may be compromised not only by equipment failure but also be(remove) due to the climbers' misjudgment of the situation. We present and analyze experimental data collected from lead climber falls on an actual climbing route, recording all positions prior to and after the fall as well as climber and belayer acceleration data. Two test series were performed, varying the fall height or the belayer mass, respectively. We find substantially larger total fall height, in particular for lighter belayers, even for relatively short falls. The major mechanisms to dissipate the energy are the energy losses when accelerating the belayer of the ground and the friction force in the loaded carabiner. We(avoid using subjective) The study conclude with recommendations for best practice under various conditions for a safe climbing experience.

Keywords:

Rock climbing safety; Belaying best practice; Rock climbing physics;
Energy conservation

1. INTRODUCTION

1.1.A Brief History of Climbing Safety

Initially, when rock climbing developed in the 19th century, safety was limited by a lack of equipment. The use of a rope served mostly the safety of the person who followed while the leading person would essentially climb unprotected. The quality of ropes was questionable and equipment to build anchors was very limited. From around 1910 on, the use of pitons, metallic pins hammered into cracks for anchoring, and carabiners to connect those with the rope allowed some basic belaying, including the lead climber [1,2,3]. Still, the equipment was not qualified by any strength tests and ropes were produced from natural fibers. The techniques used to stop a climber's fall were based on the rope slung around the body of the belayer and bare hands' strength. Had today's belay techniques been used, the brake forces would have exceeded the limited strength of the rope and other equipment. Nevertheless, the difficulty and boldness of new routes advanced to new levels. Only since the development of modern kernmantel nylon ropes in 1953 [2,4] combined with belay techniques based on the Sticht brake (similar to more recent devices like the Black Diamond ATC and other "tubes") or the HMS (Munter) hitch along with the use of climbing harnesses allowed the climbing sport to approach

safety levels that allowed to reliably stop a fall of a lead climber [5,6,7]. This enabled a more playful approach to rock climbing and a dynamic development in difficulty of new climbing routes ensued. A more detailed presentation of the development of the climbing sport can be found in [2].

1.2. Modern Sport Climbing

Today, many climbing routes are permanently equipped with regularly spaced bolts to which the rope can be clipped with carabiners. Varying local ethics often impose restrictions or entirely bans on the use of such permanent bolts, which to some extent can be remedied by the use of mobile (non-permanent) wedge-shaped stoppers or camming devices which have a lower braking strength than bolts [8] and require much experience for proper placement. When used properly, these mobile protection pieces still offer reliable protection provided the impact forces are limited to a value below their breaking strength. Parallel to outdoor climbing on natural cliffs, indoor climbing gyms offer a sheltered variety of the sport with all routes pre-equipped with bolts. All belay devices until 1991 followed the principle of dynamic belaying, providing a limited braking force and friction-controlled slippage of the rope once that limit is reached [9]. This enabled a soft stopping of the falling climber, thus limiting the peak forces on the anchors.

For indoor and on short outdoor routes starting off the ground it had become customary for the belayer to not self-anchor and just act as a counterweight to the falling lead climber. In 1991 Petzl introduced the Grigri, a semiautomatic belay device which under practical conditions locks off the rope in case of a sudden load. This greatly reduces risks due to belayer inattention while eliminating the device-provided dynamic belaying characteristics. The dynamics to ensure a soft catch of the falling climber is now provided by the belayer being pulled upward (to some extent this does also happen with a dynamic belay device, depending on climber / belayer masses, duration of the impact and belay device used). Currently, both types (dynamic and auto-locking) belay devices are in use. All climbing equipment is available with test certification [10] and material failure has become very rare.

1.3. Safety Recommendations and Testing

Safety considerations and recommendations mostly focus on the proper use of the equipment [11,12,13,14]. Such recommendations are based on observations and analysis of critical situations and accidents [15, 16], laboratory tests using static masses as fall weights [17,18], theoretical modeling [19,20,21,22,23,24] and observations and measurements with

actual climbers. While real-world falls are distinctly different from those using rigid metal masses due to the dynamics of a human body [25], systematic data collection has rarely been published except for the work of H. Mägdefrau [26] and a few videos [27]. The former measured forces on climber and belayer for a variety of fall situations and focuses on the potential harm for the climber for various harness geometries and fall situations. The latter offer force measurements for 3 fall experiments with relatively complex fall situations. Most of the theoretical modeling is describing the elastic and dissipative rope properties upon impact in an otherwise static scenario. This applies perfectly to experiments such as the standardized rope testing. One recent theoretical paper however stands out, describing exactly the real-world situation practiced when climbing off the ground, indoor or outdoor, with a static belay device [23]. In this publication the equation of motion is solved, and formulas developed for the peak forces and the length of the falls.

1.4. Scope of this study

With climbing equipment having become quite reliable and increasing frequency of lead climber falls, it is essential to analyze associated risks

and provide guidance for safe practice. Here we provide a systematic series of practical tests of lead climber falls with variation of just a single fall parameter for each series. The tests were performed with normal climbing safety gear on a regular climbing route. Two series of tests were performed, one with a variation of the free fall height, the other by varying the mass of the belayer. All tests were performed using a quasi-static belay device (Petzl Grigri) with the dynamic braking provided by the elasticity of the rope, the human body and the upwards(remove) motion of the belayer. The experimental results are compared against the model and equations presented in [23]. The purpose of this study is to determine safe belaying conditions and their limits as a function of fall height and belayer to climber mass ratio.

2. MATERIALS AND METHODS

The measurements were performed in the Sandrock [28] climbing area in Alabama, USA. A route equipped with expansion bolts for protection was selected for the tests. The route has an overhanging start leading to a vertical section (figure 1). The bolt at the base of this(replace with the) vertical

section was used to catch the falls with the carabiner catching the fall located 6.40 m above ground. The falls started in the vertical section, in line with the bolt catching the fall, and ended in the overhanging part. ~~This~~The set-up allowed for almost 1-dimensional falls with negligible pendulum motion or impact of the falling climber against the rock.

The rope was clipped into one other bolt in the overhanging section to slightly re-direct the belayer end of the rope to keep it out of the way of the falling climber. The overhanging lower part of the

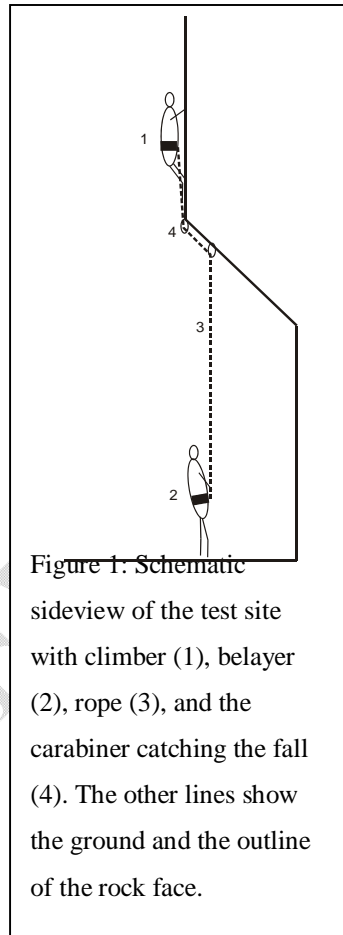


Figure 1. Schematic sideview of the test site with climber (1), belayer (2), rope (3), and the carabiner catching the fall (4). The other lines show the ground and the outline of the rock face.

route gave the belayer ample head space to be lifted off the ground without interfering with the rock. The belayer was positioned vertically below the re-directing bolt to allow for a near 1-dimensional motion similar to that of the climber. A tape measure installed parallel to the rope allowed to record starting and end positions of climber and belayer. The positions were taken at the point where the tie-in knot of the rope meets the harness. Prior to each fall all slack was removed from the rope in order to have a well-defined rope length. The rope was a typical used single rope (Edelrid Cobra, 10.3 mm diameter; impact force 9.1 kN). The carabiner to catch the falls was a Petzl

Vulcan with 40 kN closed-gate strength and 6 mm radius of curvature in the rope contact area. A Petzl Grigri was used as belaying device which provides a near static stopping action (verified during experiments using a marker on the rope), except for the lifting of the belayer who was not anchored. Climber and belayer masses, including all gear, were recorded prior to experiments.

Mobile acceleration sensors were attached to both climber and belayer and data were transferred via Bluetooth to tablets to record and save the data. Extensive prior lab tests were used to select the force sensor and its point of attachment to the body. We found it essential to use a sensor with a small dimensions and mass, rigidly attached to the body near the center of mass of the climber. The sensor of choice was the Pocketlab Voyager [29] with a mass of 17 g and dimensions of 40 x 40 x 15 mm³, attached on the side of climber and belayer at the height of the hip bone. The data analysis is based on conservation of energy to calculate the climber velocity at the beginning of the braking phase, conservation of momentum to estimate the velocities after the belayer acceleration and Newton's Laws to calculate forces [23].

The fall tests were performed with systematic variations of the height of the fall and the mass of the belayer.

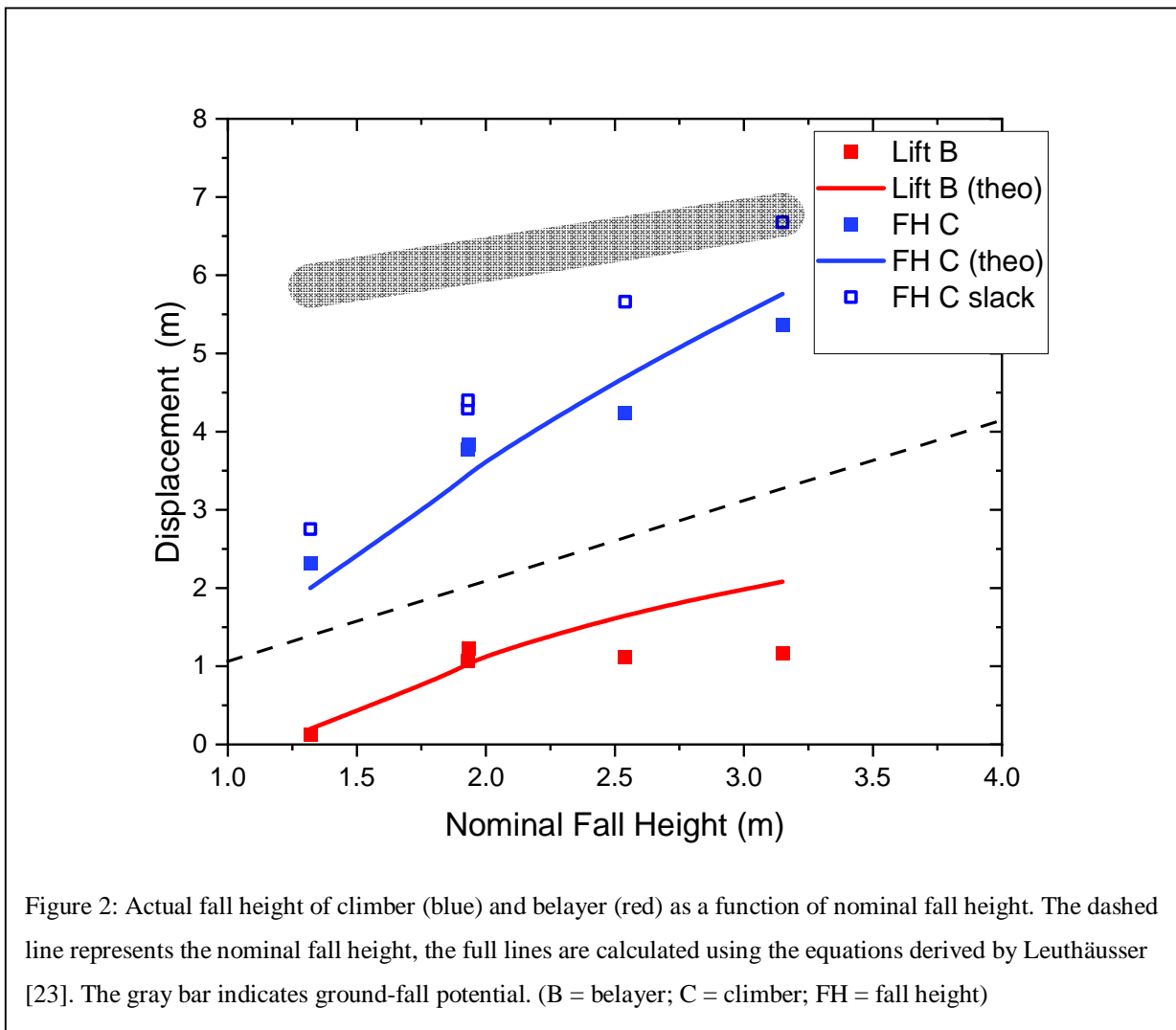
UNDER PEER REVIEW

3. RESULTS AND DISCUSSION

3.1. Variation of the Fall Height

UNDER PEER REVIEW

The first series of tests was performed under variation of the height of the



fall, with all other parameters kept constant. Climber and belayer were of similar mass (~ 70 kg) with the belayer 9% lighter than the climber. The fall height was varied from a nominal length of 1.3 m to 3.2 m. The corresponding fall factor values (length of free fall divided by total rope length loaded) varied between 0.23 and 0.47. Just prior to each fall all rope

slack was removed in order to have defined lengths for both rope and the free fall part.

Figure 2 shows the actual total fall height of the climber and the uplift of the belayer as a function of the nominal fall height. These data were taken at the end of each fall thus representing the static rope elongation, not the larger dynamic stretch during the stopping process. The dashed line shows the nominal fall height for comparison. Obviously, the total fall height is considerably larger than the nominal by about a factor 1.8. Contributing factors are the uplift of the belayer which was negligible for the shortest fall and contributed about 1 m for longer falls (the tests were explicitly performed with a passive belayer, not deliberately jumping upwards to lower the impact). The remainder comes from the elasticity of the rope which stretched about 10%. The belaying device was a Grigri which did not allow any noticeable rope slippage. We find very good agreement with calculated values using the equations derived by Leuthäusser [23, 30] (full lines in figure 2). Would the fall tests have been performed with a realistic amount of slack in the rope the resulting fall height would have been accordingly longer. We used the equations from reference [23] to calculate the climber fall allowing 0.5 m of slack (open blue symbols in figure 2). The fall length increases considerably beyond the mere 0.5 m of added rope length due to

the increased input of potential energy. While 0.5 m rope slack could be considered a bare minimum for undisruptive rope handling by the belayer, this would have led already to ground-fall potential for the longest fall. To put it into perspective, this would be for a relatively small fall at the 3rd bolt in a rather well-protected climb. In practice, climbers can often be observed to allow slack far in excess of this, sometimes combined with improper sideways positioning of the belayer, or with extra rope paid out to clip an overhead bolt. This would lead to ground contact of the lead climber even for the shortest fall and to an almost unimpeded ground fall for the longer falls. This is somewhat disconcerting for a situation with the 3rd bolt of the route clipped (6.7 m above ground) and trying to clip the 4th bolt. This enhanced ground-fall risk is typically associated with falls into the first bolt when attempting to clip the second bolt of a route. Depending on bolt positions, this risk clearly extends far beyond the 1st and even the 2nd bolt.

Figure 3 shows the maximum rope forces exerted onto the climber, the belayer and the bolt, respectively, during the stopping of the fall as a function of nominal fall height. The experimental values were calculated by multiplying the measured maximum acceleration values with the climber mass. The force on the bolt is found by adding the forces acting on climber and belayer. The highest rope force acting on the climber observed for the longest fall equals about 4 times his weight, thus resulting in a peak acceleration of 3g. The rope forces on the belayer are much lower, the

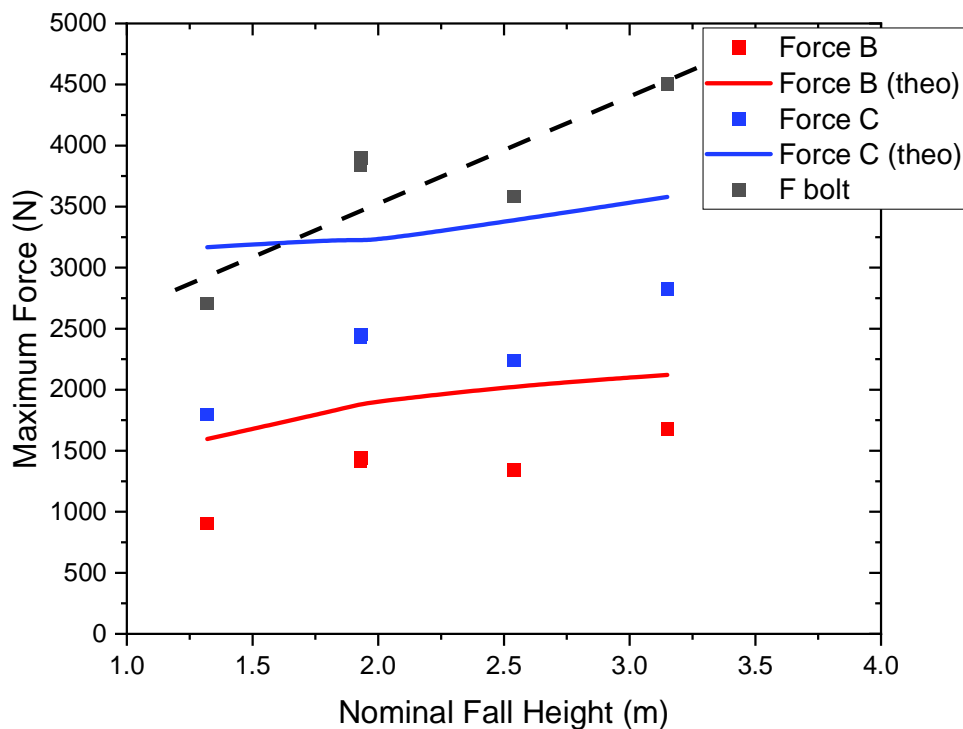


Figure 3: Maximum forces the rope exerts on climber, belayer and bolt, respectively, as a function of nominal fall height. The full lines are calculated using the equations derived by Leuthäusser [23].

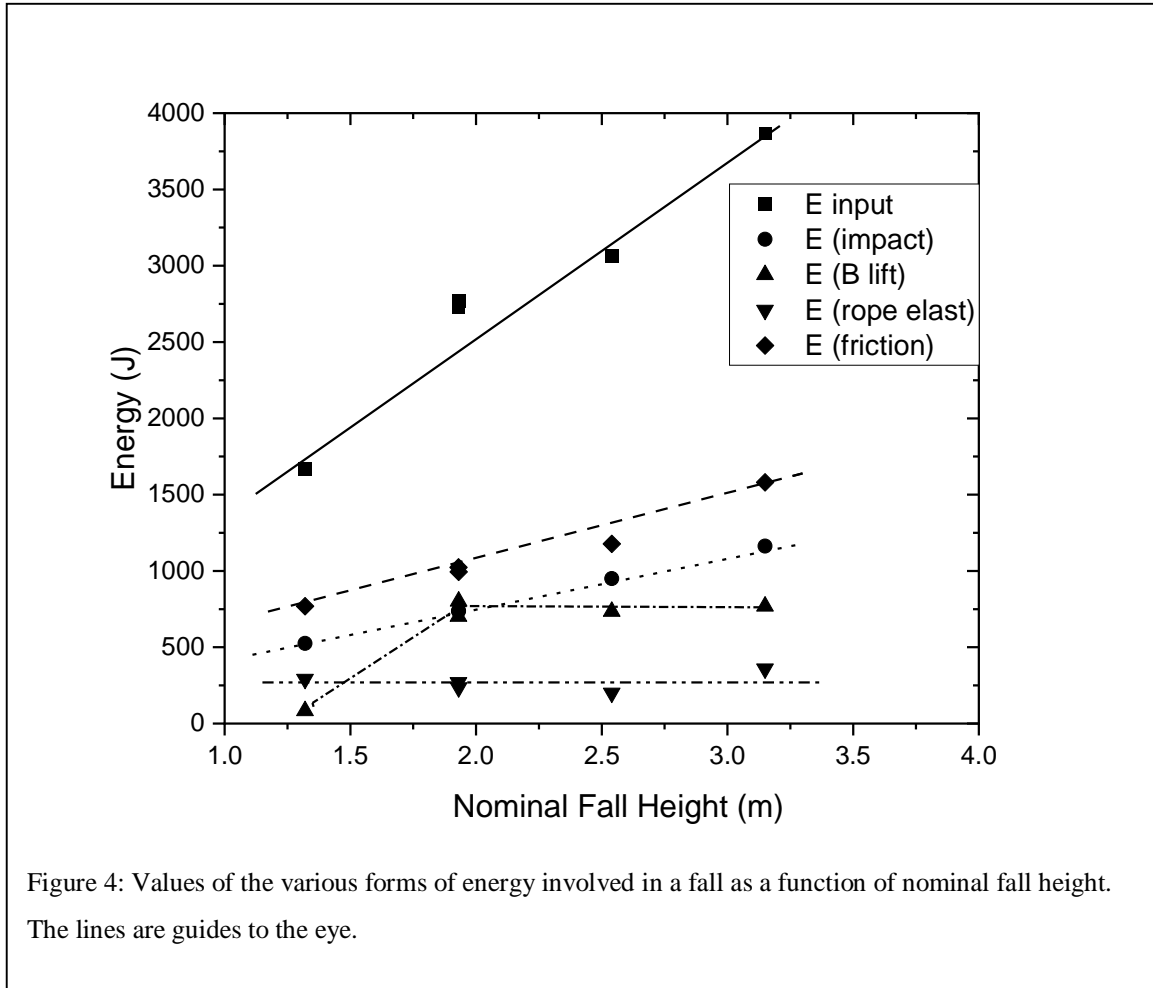
difference coming from the additional friction of the rope being redirected at the carabiner which catches the fall. This carabiner friction acts essentially as a brake force multiplier, the value of which can be computed by dividing the peak rope force acting on the climber

by that acting on the belayer. This brake force multiplier factor is about 1.7 – 1.9 for all falls, consistent with other reports [24**Error! Bookmark not defined.**]. Comparing with the work of Leuthäusser [23] we find that this theory slightly overestimates the forces on both, climber and belayer, but the trend is comparable. Larger fall heights were not accessible at the chosen test site so an extrapolation of the experimental data is speculative. The theory clearly suggests further increasing rope forces with increasing fall height. This differs from the claim made in reference [26] suggesting the force on the climber to reach a plateau, however, without providing an argument or reference. The impact of the fall can certainly be softened by using additional dynamic elements such as a belayer who is deliberately jumping off the ground or using a dynamic belaying device instead of a Grigri. However, this would further increase the fall height.

The total force acting on the bolt does not exceed 4.5 kN, a safe value for a properly placed bolt and also for most removable gear, if properly placed. Larger falls will potentially approach the load limits of some gear, so caution

on the belayer side is required. Another aspect is the energy dissipation which originally stems from the change of potential gravitational energy of the falling climber. When the fall is being caught, after an initial stretch of the rope, the belayer will be lifted up, accelerating him to the speed of the climber. While other channels of energy dissipation will set in partially concurrently, we model this initial acceleration of the belayer as an inelastic collision using conservation of momentum, resulting in a common final speed. Then the initial potential energy of the free fall, turned kinetic energy, gets reduced upon acceleration of the belayer to $m_c / (m_c + m_b)$, with m_c and m_b the masses of climber and belayer, respectively. Thus, the lost kinetic energy due to this inelastic collision then is $m_b / (m_c + m_b)$. The energy stored in the final, static elongation of the rope is calculated using Hooke's Law. Part of the energy is imparted onto the belayer, lifting him up thus increasing his potential energy. The remaining energy losses will include all frictional effects, in particular the rope slipping through the carabiner that catches the fall [24**Error! Bookmark not defined.**]. A direct calculation of its value is elusive because of the time-dependent forces. Therefore, it is calculated using conservation of energy. In summary, the initial energy (potential energy change of the climber) gets divided up into a) potential

energy increase (uplift) of the belayer, b) final elastic energy of the stretched rope, c)



energy loss due to acceleration of the belayer (collision), and d) friction losses in the carabiner.

Figure 4 shows these various forms of energy as a function of nominal fall height. The two dominating factors of energy dissipation due to accelerating the belayer and friction of the rope running through the carabiner that catches the fall, it essentially assumes the role of a dynamic brake. The uplift of the belayer is negligible for short falls (the belayer

remains near the ground) but plays a role for longer falls. The elastic elongation of the rope is not an important factor.

3.2. Variation of the Belayer Mass

In the following experiments the climber mass ($m_c = 73.7$ kg) and the nominal fall height ($h_n = 2.54$ m) were kept constant, while varying the mass of the belayer. An increase of belayer mass clearly reduces the uplift of the belayer and consequently the actual fall height (figure 5). While a fall caught by a belayer with

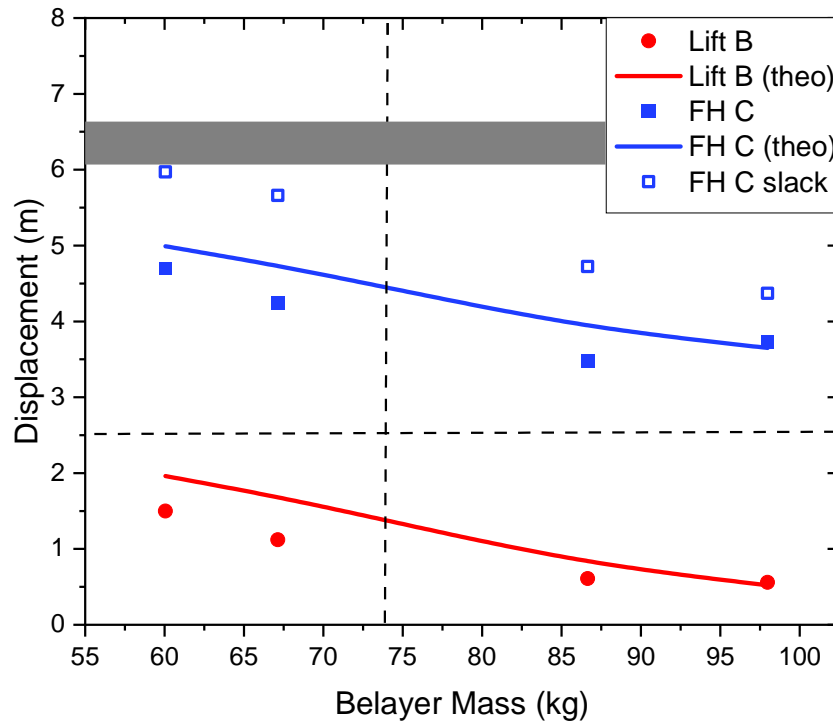
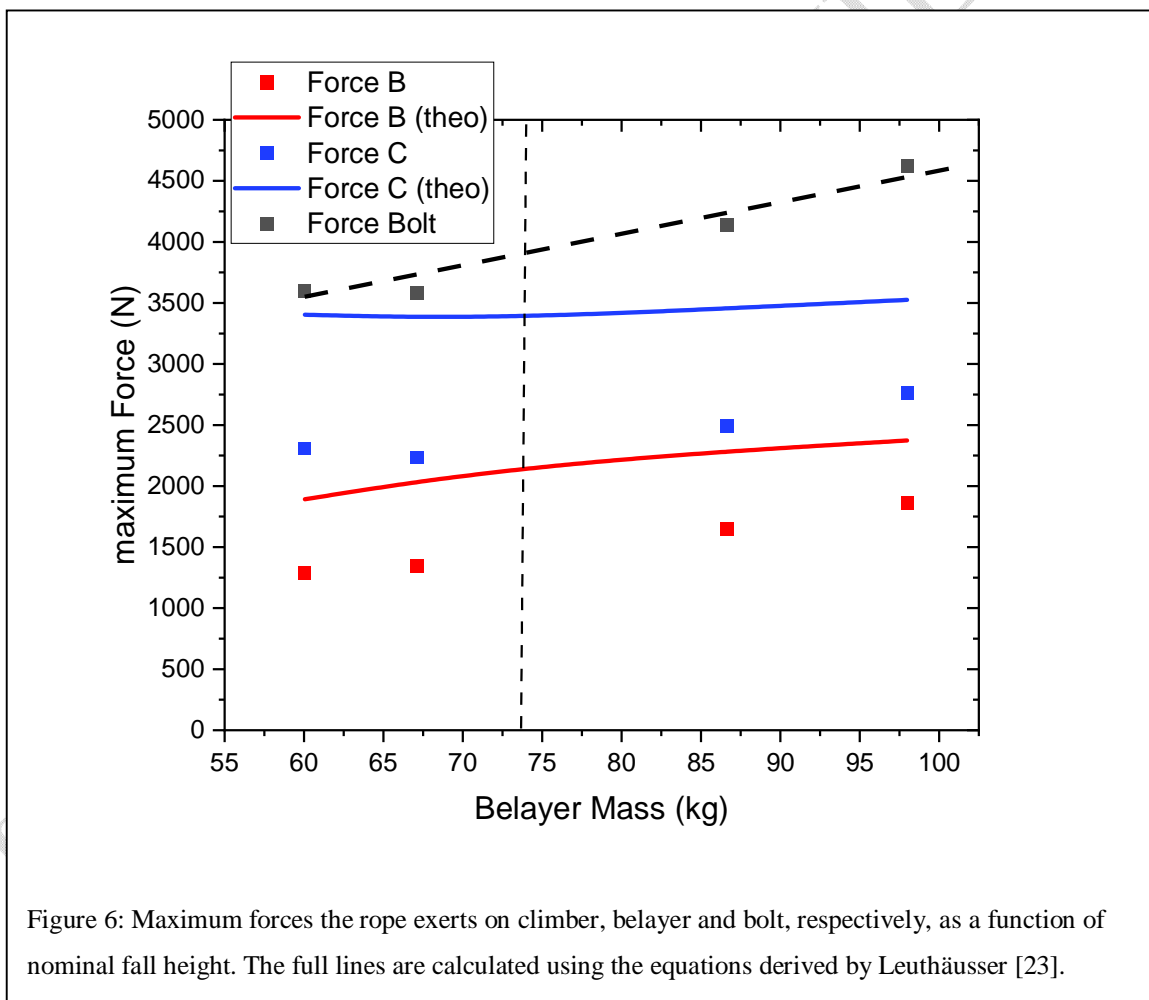


Figure 5: Vertical displacement of climber (actual fall height) and belayer as a function of the belayer mass. Nominal fall height ($h_n = 2.54$ m) and climber mass ($m_c = 73.7$ kg) were kept constant. The vertical dashed line indicates m_c , the horizontal dashed line h_n . The full lines are calculated using the equations derived by Leuthäusser [23]. The gray bar indicates ground-fall potential.

about 80% of m_c ends up being almost twice the nominal fall height, a belayer with 130% m_c limits this to about 40% increased fall height (with about 25% increased fall height being due to rope stretch alone). Thus without intentional upwards motion of the belayer catching a fall with a static belay device (Grigri) causes the braking to become essentially static at a belayer mass of $\sim 150\%$ m_c . a static catch of the fall should be avoided by either using a dynamic belay device or the belayer actively jumping upwards

to soften the catch. On the other hand, the absolute lower limit of the belayer mass is 55% of m_c at which point the friction in the carabiner will no longer stop the fall. Practically, climbers should take steps to avoid approaching this limit. Calculating total fall height values for a realistic, but still small, value of 0.5 m rope slack (open, blue data point in figure 5) shows that even a belayer with 80% mass of the climber can barely prevent a ground-fall.



The corresponding forces (figure 6) the rope exerts on climber, belayer, and the bolt increase by 20 - 30% when varying the belayer mass from 80% m_c to 130% m_c . The highest values of the force on the climber correspond to

about 4g, 1g to balance the weight and 3g actual deceleration. The theoretical modeling based on [23] again systematically overestimates the forces, but the trend matches very well the experimental data. Potential explanations will be discussed in the conclusions. Lighter belayers would need extra mass, self-anchoring or use additional friction devices like the Edelrid Ohm to be able to control both, the fall height and their own motion, in order to prevent a collision with the rock or falling climber that may result in injury. Heavier belayers would either have to resort to deliberately supporting their upwards motion or use a dynamic belay device. In the latter case, the belay device would need to have a brake force smaller than the force acting on the belayer to allow rope slippage. Ref. [17**Error! Bookmark not defined.**] experimentally analyses the brake dynamics of several models. With the actual brake force depending on hand strength, this can be varied provided the belayer is well-trained.

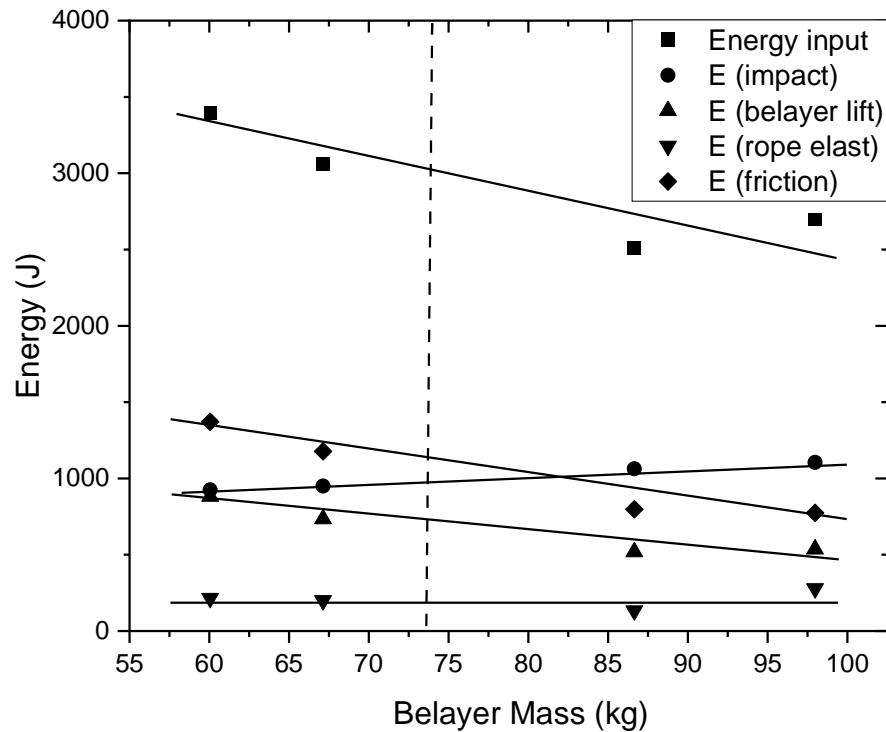


Figure 7: Values of the various forms of energy involved in a fall as a function of belayer mass. The lines are guides to the eye.

The energy contributions are shown in figure 7 as a function of belayer mass. The total energy decreases with increasing belayer mass because of the reduced actual fall height. For all scenarios the two most important channels for energy dissipation are friction in the loaded carabiner and energy spent on accelerating the belayer (“collision” energy) with the latter dominating for heavier climbers. For lighter belayers the potential energy gain due to uplifting the belayer also comes into play, but this is counterproductive as it results in even larger input energies.

4. CONCLUSIONS

Using real-world data recorded during lead climber falls give a good insight into the limiting factors for catching a fall. The actual fall height is found to be in good agreement with theoretical modeling and it easily exceeds what most climbers would anticipate not only due to stretch of the rope but also caused by lifting the belayer and slack in the rope. This was evidenced by observing a party of climbers taking a fall that ended with the climber's feet on the ground and the belayer nearly 1.5 m in the air. The ground-fall risk is normally considered around the first or second bolt of a climb. Our tests were conducted at the third bolt of a well-protected route and still, a ground-fall is a very present risk. The tests reported here were done without slack and using a static belaying device and still the permissible fall height was quite limited. In particular, a lightweight belayer (< 80% of the climber mass) would be pulled up considerably while enlarging the climber's fall height, thus increasing the risk of collision with overhead rock structures, with the falling climber as well as the climber hitting the ground. Lighter belayers would need to self-anchor and utilize additional friction devices like the Edelrid Ohm. The use of a dynamic belaying device might keep the belayer closer to ground, but at the expense of a longer fall for the climber. Belayers considerably heavier than the

climber should use a dynamic belaying device [17] or actively jump upwards to soften the impact. The general problem is that the desire to give universal advice for a climber/belayer team gets quickly compromised when the climb proceeds with the rope accumulating friction when zigzagging through multiple carabiners. Adding another 180 degree accumulated rope redirection has essentially the effect of doubling the effective mass of the belayer. Thus, a belayer who for the first part of a route was rather “underweight” and had to self-anchor or rely on an Edelrid Ohm device would fail to provide a soft enough catch to prevent injury if the climber fell higher up in the climb. Practically, streamlined textbook advice cannot substitute for experience of taking and catching falls in various situations.

The forces we find are somewhat lower than predicted by theory [23]. This work found better agreement with experimental data reported by [27] but those experiments are more convoluted and harder to relate to the theoretical model. Also, the forces they report might be the rope force and, if so, they would be overestimated by the theory by about the same amount as we find here. Reference [23] considers force reductions due to the softness of the human body but dismisses those based on elasticity values found for shock-wave propagation through a human body [31]. The relatively slow force variations for climbing falls might, however, allow other reactions like

movement of limbs which, for smaller falls, could indeed soften the impact forces. This was also reported experimentally in [32]. In all cases the maximum load on the anchor remained below rated strength of bolts and standard mobile gear, keeping in mind that for mobile gear not only the strength of the gear itself but also that of the surrounding rock needs to be considered. Larger falls will lead to forces approaching the breaking strength of some gear, thus requiring strategies to soften the catch. The analysis of the energy dissipation channels shows the dominance of two factors, the energy loss due to acceleration of the belayer and the friction in the carabiner that catches the fall. The latter actually is the only remaining brake when using a static belay device. The importance of this friction in the carabiner contact was also noticed in prior work [24**Error! Bookmark not defined.**,17]. Without it, dynamic belay devices with a low brake force would not even statically hold the weight of a climber [17], let alone stop a fall.

In summary, we have analyzed a comprehensive and systematic set of experiments of climber falls stopped with a static belay device and a not anchored belayer. The actual fall distances show that ground fall risk persists much higher up into a climb than typically discussed. Advice to climbers should not be attempted to be cast into a few simple rules, the situation

simply depends on too many factors. Falling as a lead climber and catching falls should be a subject of practice under qualified supervision. Further experiments are planned to study the effect of dynamic belaying and belayers positioned away from the vertical line. The latter often is necessary to stay out of the way of the climber but sometimes is observed to be done to an extent which does not seem safe.

Declarations

Ethical approval

All procedures performed in studies involving human participants were in accordance with the ethical standards of the institutional and/or national research committee and with the 1964 Helsinki declaration and its later amendments or comparable ethical standards. IRB approval was obtained under # 20-03-3355 (The University of Alabama).

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