

# Determination of energy dissipation in stepped spillway with fully developed hydraulic jump by a ball that bounced down through it.

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## ABSTRACT

In this paper, the authors reviewed the characteristics of a ball that descended one step each bounce of a flight of stairs and drew an analogy with a stepped spillway with a fully developed hydraulic jump. They used this concept with equations of motion, classical hydraulic jump, and Chanson's model for energy dissipation in the stepped spillway to formulate a mechanistic model for energy dissipation rates. The range of Froude numbers used was between 5 and 16. They used data from previous researchers' publications on drop structures to verify the new model and data sets generated compared well with the measured data sets. Results showed that energy dissipation rates along the stepped spillway increased with the increasing number of steps but decreased with increasing discharges. It also showed the optimal number of steps for 100% energy dissipation.

*Keywords: Energy Dissipation, Stepped spillway, Nappe flow, jet length, minimum chute length, hydraulic jump length*

## 1.0 INTRODUCTION

In this paper, the authors reviewed the characteristics of nappe flow with a fully developed hydraulic jump on the stepped chute and drew an analogy with a ball that descended one step each bounce. They used the concept to develop a mechanistic model for the rates of energy dissipation for nappe flow with a hydraulic jump in a stepped spillway.

Nearly one-third of dams built in Europe and North America were equipped with a stepped cascade during the 19th century (Stephenson, 1991). More recently, the 1980s and 1990s were marked by renewed interest in stepped spillway design (Chanson, 2001).

Most structures were steep chutes for gravity dams operating in a skimming flow regime. For relatively low flow rates, the waters cascade down a stepped spillway as a succession of free-falling nappes.

Researchers like Chanson (1994) classified flows through stepped spillways into three regimes a) Nappe flow regime with a fully developed hydraulic jump, b) Nappe flow regime with a partial flow regime, and c) skimming flow regime. The nappe flow regime is a series of plunges from one step to another with the formation of nappe at each drop. This type of flow is approximated by a series of single-step drop structures (Chamani and Rajaratnam, 1994; Chanson, 1993). The flow leaves the step as a free-falling jet and impinges on the tread of the next step. Energy dissipation occurs by the jet breakup, jet mixing on the steps, and formation of a partially or fully developed hydraulic jump on the steps (Chanson 1994; Rajaratnam 1990). Essery and Homer (1978) and Peyras et al.(1992) described a fully developed hydraulic jump (Fig. 1) as an isolated nappe flow. The flow passes through critical

50 depths at the brinks of the steps, forming supercritical free-falling jets, and returns to  
51 subcritical flow downstream of the jump.

52 In a nappe flow with a fully developed hydraulic jump the head loss at any intermediary step  
53 equals the step height. Hence, the total head loss,  $\Delta H$ , along the spillway is the difference  
54 between the maximum head available,  $H_{max}$ , and the residual head,  $H_e$ , at the bottom of the  
55 spillway  $H_1$  (Chanson, 1994). These energy losses could be calculated using equations [1] or  
56 [2].  
57

58 For horizontal uniform steps, the flow depth at the edges is  $d_b = 0.715 * d_c$ , where  $d_c$  is the  
59 critical flow depth (Rouse, 1936).  
60

61 Nappe flow with an established hydraulic jump (Fig. 1) usually occurs from small discharges  
62 with shallow flow depths. The flow cascades over steps, with the formation of supercritical  
63 flow at the edges of the steps, and returns to subcritical flow downstream of the jump.

64 In the nappe flow regime, a sequence of drops from one step to the next lower step occurs  
65 with the formation of a hydraulic jump on each step. This flow type is like a sequence of  
66 separate drop structures (Chamani and Rajaratnam 1994; Chanson 1993).

67 The water flows over one step of the spillway to the next lower step with energy loss  
68 occurring from: a) the disintegration of the jet in the air and b) the mixing of the flow on the  
69 steps, with or without the development of hydraulic jump, on the step (Chanson, 1994;  
70 Rajaratnam, 1990). Equations [1] and [2] can be adopted to calculate the rates of energy  
71 losses in stepped spillway.  
72

$$\frac{\Delta H}{H_{max}} = 1 - \frac{\frac{d_1}{d_c} + \frac{1}{2} \left( \frac{d_c}{d_1} \right)^2}{\frac{H_{max}}{d_c} + \frac{3}{2}} \quad [1]$$

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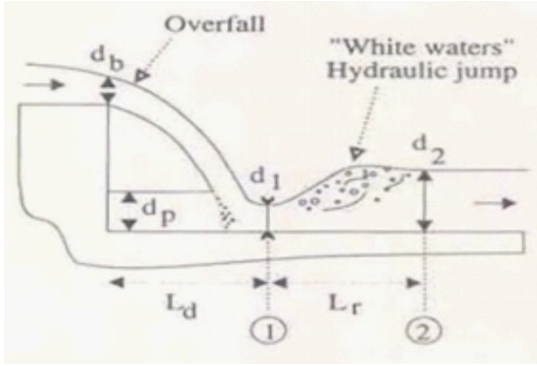
74 Where  $d_1$  is the water depth at impact,  $d_c$  is the critical water depth, and  $H_{dam}$  is the dam  
75 height,  $\Delta H$  is the energy loss,  $H_{max}$  is the maximum available energy,  $h$  is the height of the  
76 spillway step.

77 Chanson (1994) later expressed this equation in terms of the spillway step height, the critical  
78 flow depth and the dam height as:  
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$$\frac{\Delta H}{H_{max}} = 1 - \left[ \frac{0.54 \left( \frac{d_c}{h} \right)^{0.275} + \frac{3.43}{2} \left( \frac{d_c}{h} \right)^{-0.55}}{\frac{H_{max}}{d_c} + \frac{3}{2}} \right] \quad [2]$$

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81 Nappe flow with an established hydraulic jump (Fig. 1) usually arises from small discharges  
82 with shallow flow depths. It flows over the steps with the formation of supercritical flow at the  
83 edges of the steps and returns to subcritical flow downstream of the jump.  
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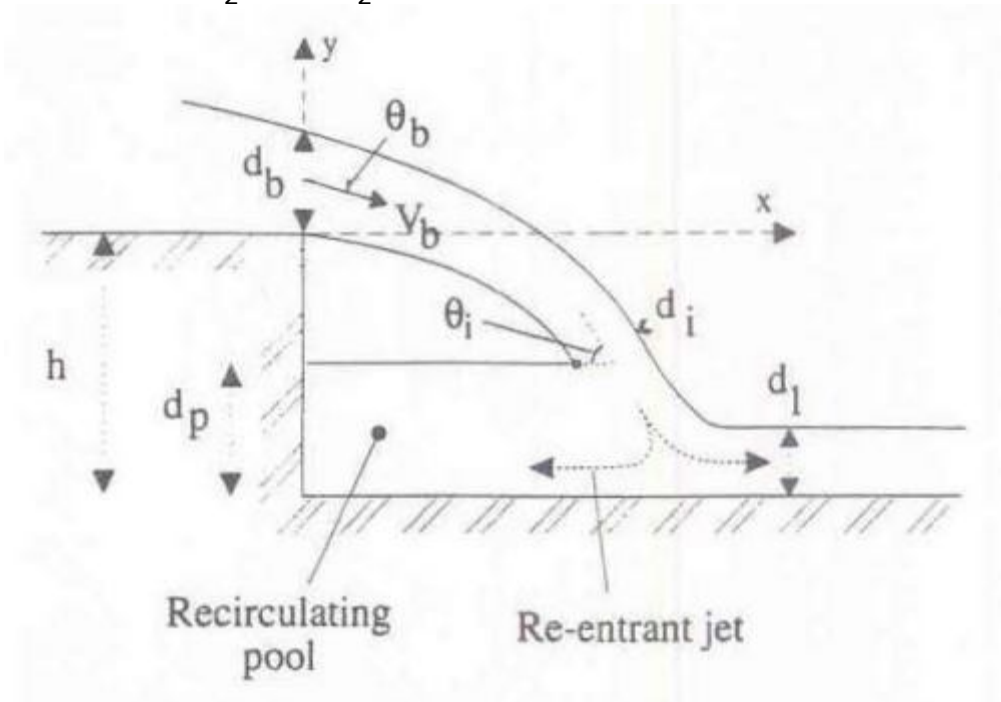


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**Fig. 1. Nappes flow trajectory at a fall structure**

Estimate of the transition nappes flows on a stepped channel considering a nappes flows down in a single-step structure, the air cavity below the falling nappes will vanish when the circulating pool of water fills the total step cavity (Fig.1). The recirculating pool of water is essential as the accompanying pressure force offers a force parallel to the step surface, which is required to change the jet direction from an angle to a horizontal. For an aerated nappes, the momentum equation resolved along the step surface is:

$$\frac{1}{2}\rho_w g d_p^2 - \frac{1}{2}\rho_w g d_1^2 = \rho_w q_w (V_1 - V_i \cos \theta_i) \quad [3]$$



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**Fig. 2. Nappes flow trajectory at a fall structure**

where  
 $d_p$  = pool peak,  
 $d_1$  = flow depth downstream of the jet impact,  
 $V_i$  = related velocity,  
 $V_1$  = impact velocity,

104  $d_i$  = flow depth at the step edge,  
 105  $V_b$  = flow velocity at the edge,  
 106  $\vartheta_i$  = jet impact angle.  
 107  $\vartheta_b$  = the original angle (Figure 2.6)  
 108 is the Froude number at the step brink of the streamlines with the horizontal is  
 109  $\vartheta_i$  = jet impact angle.

110  
 111 Equation [3] presumes that the brinks of the nappe have not broken down into spray,  
 112 disregards the shear forces on the surfaces, and considers a hydrostatic pressure  
 113 distribution at section 1 and the vertical face of the steps (Fig. 2).  
 114 The impact flow conditions can be inferred from simple trajectory calculations (Chanson,  
 115 1995):  
 116 where

$$\frac{d_i}{d_b} = \left(1 + \frac{1}{Fr_b^2}\right)^{-1/2} \quad [4]$$

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$$\frac{V_i}{V_b} = \sqrt{1 + \frac{1}{Fr_b^2}} \quad [5]$$

118

$$\frac{\cos\theta_i}{\cos\theta_b} = \left(1 + \frac{1}{Fr_b^2}\right)^{-1/2} \quad [6]$$

119 Where

120  $d_b$  = flow depth at the step edge,  
 121  $v_b$  = flow velocity at the edge,  
 122  $\theta_b$  = original angle of the streamlines with the horizontal (Fig. 2),  
 123  $Fr_b$  = Froude number at the step brink.

124 At the jet crash on the step, and presuming that the velocity of flow into the control volume is  
 125 the same as the one out of it, the momentum equation Eq. 2.1 yields:  
 126

$$\frac{d_p}{d_i} = \sqrt{1 + \frac{2v_i^2}{gd_i} (-\cos\theta_i)} \quad [7]$$

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128 White (1943) and Rouse (1955) analyzed a unit-step drop structure (Fig 2). For a flat step,  
 129 the flow situations close to the steps change from the sub-critical flow to critical flow at a  
 130 definite sector upstream of the brink (Chanson, 1994)

131 Where  $d_b$ , is given by ROUSE (1936) as

132

$$d_b = 0.715d_c \quad [8]$$

133

134 where  
 135  $d_c$  = critical flow depth.

136 Downstream of the brink of the step, Rand (1955) linked the following nappe geometry

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$$\frac{d_1}{h} = 0.54 \left(\frac{d_c}{h}\right)^{1.275} \quad [9]$$

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$$\frac{d_2}{h} = 1.66 \left(\frac{d_c}{h}\right)^{0.81} \quad [10]$$

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$$\frac{d_p}{h} = \left(\frac{d_c}{h}\right)^{0.66} \quad [11]$$

141 Resulting to

$$\frac{L_d}{h} = 4.30 \left(\frac{d_c}{h}\right)^{0.81} \quad [12]$$

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$$\frac{d_1}{d_c} = \frac{2^{\frac{1}{2}}}{\frac{3}{2^{\frac{3}{2}}} + \sqrt{\frac{3}{2} + \frac{h}{d_c}}} \text{White(1955)} [13]$$

where  $d_1$ ,  $d_2$ ,  $d_p$ ,  $d_c$ ,  $L_d$ ,  $L_r$ ,  $h$  are the flow depth at sector 1, flow depth at sector 2, are the distance from the vertical façade of the step to the point of contact, the roller length of a completely established hydraulic jump, and length of a flat step, respectively (Fig. 1) (Chanson, 1994).

The flow depth,  $d_2$ , and the entire head at the sector 2 (Fig.1) are linked by the traditional hydraulic jump equation (Chanson)

$$\frac{d_2}{d_1} = \frac{1}{2} \left\{ \sqrt{1 + 8Fr_1^2} - 1 \right\} [14]$$

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Where  $L_r$ ,  $d_1$ , and  $Fr_1$  are the roller length of a completely established hydraulic jump, The depth of flow and the Froude Number respectively directly upstream of the jump is given as;

$$Fr_1 = \frac{q_w}{\sqrt{gd_1}} [15]$$

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## 1.2 Conditions for nappe flow regime

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If the length of the descent plus the length of the roller,  $L_r$ , is smaller than the length of the flat step,  $l$ , an established hydraulic jump occurs (Fig 1) Chanson (1994). Merging equations (9) and (10) a situation for an established napped flow is obtained: a nappe flow regime with a hydraulic jump takes place for flow rates that are less than a critical value defined by Chanson (1994) as:

$$\left(\frac{d_c}{h}\right)_{char} = 0.0916 \left(\frac{h}{l}\right)^{-1.276} [16]$$

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Where the relationship (Eq[16]) was found for  $0.2 \leq h/l \leq 6$

And  $l$  is the step length. Nappe flow for completely established hydraulic jump takes place for

$$d_c/h < \left(\frac{d_c}{h}\right)_{char} \quad [17]$$

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Stephenson (1991) suggested from the knowledge gained from dams with stepped spillways in South Africa that the suitable condition for nappe flow settings is:

$$\tan\theta = h/l < 0.2 \quad [18]$$

173

174 For nappe flow to take place, the step flat length should be more than the water depth  
175 (Stephenson 1991, Lejeune et al. 1994)

### 176 1.3 Trajectory of a free falling ball

177 A ball released from a certain height will take equal time to drop as another ball projected  
178 horizontally from the same high, notwithstanding the different path length, as the vertical  
179 acceleration is the same. A ball thrown vertically up at twice the speed will rise four times  
180 higher, not because of  $v^2 = 2as$  (an equation rather than an “explanation”), but because it  
181 takes twice as long to stop and travels twice as fast on average. A ball thrown obliquely onto  
182 the floor without spin will bounce with spin due to the horizontal friction force on the ball. A  
183 smaller ball will spin faster than a big ball of similar mass as it has a small moment of inertia.  
184 When a falling ball first contacts a rigid horizontal surface, the bottom of the ball instantly  
185 comes to a complete stop. All balls are sufficiently flexible that the top of the ball will  
186 continue to fall even after the bottom comes to rest. The ball will therefore compress like a  
187 spring, progressively from the bottom end up as each new section comes to rest. However,  
188 the bottom end of a ball is much softer than the middle section since the cross-sectional area  
189 is smaller, being a few mm in diameter at first compression. The effect is equivalent to  
190 having a relatively stiff and heavy mass compressing down on a relatively light and soft  
191 spring. The result is that only the bottom part of a ball compresses in a low-speed collision. A  
192 large portion of the ball compresses in a high-speed impact (Rod, 2015).

193  
194 Trajectory refers to the path a falling or a rising object follows. In projectile, only acceleration  
195 due to gravity ( $g$ ), which acts in the vertical direction, is considered in the equation of motion.  
196 There is no acceleration in the horizontal direction, with the small air resistance ignored in  
197 the equation of motion. Some examples of Projectile Motion are Football, A baseball, A  
198 cricket ball, or any other object. The projectile motion consists of two parts: one in the  
199 horizontal direction of zero acceleration and the other in the vertical direction of constant  
200 acceleration due to gravity. The projectile motion is always in the form of a parabola and  
201 represented as:

$$202 \quad y = ax + bx^2 \quad [19]$$

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205  
206 Damping of motion occurs when air resistance is neglected and calculated as the square  
207 root of the ratios of maximum height after and before a bounce. The velocity of a ball with  
208 mass,  $m$ , which falls freely from a height,  $h_1$ , and hits the ground, can be given as  
209

$$210 \quad v_1 = \sqrt{2gh_1} \quad [20]$$

211  $g$  = the acceleration due to gravity.

212  
213 This is true because the potential energy,  $mgh_1$ , is completely change into kinetic energy,  
214  $mv_1^2/2$ . The velocity  $v_2$  of the ball upon bounces to a height,  $h_2$ , is given as

$$215 \quad v_2 = ev_1, \quad [21]$$

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217  
218 Where  $e$  is the coefficient of restitution and  $v_2$  is derived as

$$219 \quad v_2 = \sqrt{2gh_2} \quad [22].$$

220  
221  
222 Therefore, 
$$e = v_2/v_1 = \sqrt{h_2/h_1} \quad [23]$$

223

224 The time,  $t_1$  for the first fall,  $h_1$ , is

$$h_1 = ut_1 + gt_1^2/2 \quad [24]$$

225

226 Where  $u$  = the initial vertical velocity = 0

227

228 Simplifying,  $t_1 = \sqrt{2h_1/g}$  [25]

229

230 Similarly, the time,  $t_2$ , for the second fall,  $h_2$ , is

231

$$t_2 = \sqrt{2h_2/g} = et_1 \quad [26]$$

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233 The total time,  $T$ , from the initial time to complete rest is therefore given as

234

$$\begin{aligned} T &= t_1 + 2t_2 + 2t_3 + \dots = t_1 + 2et_1(1 + e + e^2 + \dots) \\ &= t_1 + 2et_1/(1 - e) = t_1(1 + e)/(1 - e) \end{aligned}$$

235

$$T = \sqrt{2h_1/g} (1 + e)/(1 - e) \quad [27]$$

236

237 The point to note here is that most mathematical models predict ball bounces as being  
238 infinitely, where the total time for the ball to come to rest is finite (Heck and Ellermeijer,  
239 2009).

240

241 The authors studied the trajectory of the ball that bounced down the stairs with energy loss  
242 occurring at bounce. They assumed that the ball bounces off each step and hit the steps  
243 below as a free-falling object with energy dissipation occurring on collision with the horizontal  
244 step surface.

245 For simplicity, they assumed that the coefficient of restitution ( $e$ ) was constant during the  
246 entire bounce dynamics and took on a positive value 'e' less than unity. They considered a  
247 stairway, which consisted of horizontal and vertical parts only.

248

#### 249 **1.4 Empirical modeling/Mechanistic modeling.**

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251 Although many researchers have investigated the hydraulic performance of stepped  
252 spillways, significant information gaps still exist in the guidelines for the design of stepped  
253 spillways. Most of them proposed empirical models for rates of energy loss in stepped  
254 spillways, which in most cases are applicable only for the modeled experiment under a  
255 particular operating condition. Therefore, empirical models cannot predict beyond certain  
256 operating ranges or designed experiment: Hence, the need for mechanistic models.

257 Mechanistic Modeling is adopted where the physical occurrence is approximated by  
258 considering the best essential processes and neglecting other less important effects that can  
259 complicate the problem but do not add considerably to the accuracy of the solution. It should  
260 be sufficiently close to the natural phenomenon as the flow pattern involved should not be  
261 overlooked. Yet, it should be largely simple so that the solution is amenable to reasonable  
262 analytical or numerical efforts (Taitel, 1994).

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264

## 265 **2. METHODOLOGY**

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267 The authors used the Newton Law of Motion, the classical hydraulic jump equation (Eq [5]),  
268 and the energy dissipation at nappe flow equation (Eq [9]) to develop this model. They used  
269 the law of motion to formulate the trajectory of an elastic pinball that bounced down a drop

270 structure of horizontal step,  $l$ , and height,  $h$ , (Fig 1) and later extended the concept to a flight  
271 of stairs made up of multiples of similar drop structures.

272 They assumed

- 273 i) That the ball descended one step each bounce  
274 ii) That after each bounce, it rebounded to a height,  $d_2$ ,  
275 iii) That the height,  $d_2$ , was large enough compared with the width of the step so that  
276 the impacts were effectively head-on;  
277 i) That the coefficient of restitution,  $e$ , is constant and is less than 1;  
278 ii) That air resistance is overlooked;  
279 iii) That the only acceleration in the system is the acceleration due to gravity,  $g$ ;  
280 iv) That there is no acceleration in the horizontal direction,  
281 v) that the trajectory of a freely falling ball is similar to that of the nappe flow regime,  
282 which is distinguished by a series of plunges, from one step to another with the  
283 formation of a nappe at each drop;  
284 vi) that ball passed through a critical depth at the brink of each step, forming a  
285 supercritical free-falling flow and returned to subcritical flow downstream of the jump  
286 and;  
287 vii) that the manner energy is dispersed in a nappe flow regime with a full hydraulic  
288 jump is similar to the way energy is lost with a freely falling ball descended one step  
289 with each bounce.

290 Since the collision occurred with the vertical component of the incidence velocity, the  
291 rebounded vertical velocity became 'e' times the opposite of this bound vertical velocity while  
292 the horizontal distance the ball traveled from  $d_1$  to  $d_2$  or  $x$ , became 'ut', measured from the  
293 edge of the step where bounce occurred.

294

295 The ball bounced off the step tread through the height,

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$$297 \quad d_2 - d_1 \quad [28]$$

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299 And landed on the next lower step through the height,

300

$$301 \quad d_b + h - d_1 [29]$$

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303 where  $d_b$ ,  $h$ ,  $d_1$ , and  $d_2$ , respectively are the flow depth at the brink of the horizontal step, the  
304 height of step, the depth of bounce at section 1, and the depth of bounce at section 2 (Fig 1).

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306 The coefficient of restitution denoted by 'e' - and ranges between 0 and 1 - is the ratio of the  
307 final to initial relative velocity between two objects at collision is defined as

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$$e = \sqrt{\frac{(d_2 - d_1)}{(d_b + h - d_1)}} \quad [30]$$

309

## 310 **2.1 Measured data sets for verification of the new model**

311

312 They obtained more than fifty field data from the published works on the rates of energy  
313 dissipation of drop structures by Moore (1943), Rand (1955), and Stephenson (1979) and  
314 used them to verify the new model.

315

## 316 **2.2 Formulation of model**

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318 Simplifying, Eq [3] yields

$$d_1 = \frac{(d_2 - e^2 d_b - e^2 h)}{(1 - e^2)} [31]$$

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Substituting equations [8] and [31] into equation [31] and simplifying yields

$$d_1 = \frac{\left(\frac{d_1}{2} \left\{ \sqrt{1 + 8Fr_1^2} - 1 \right\} - e^2 0.75d_c - e^2 h\right)}{(1 - e^2)(e^2 - 1)} [32]$$

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$$d_1 = \frac{-e^2(0.715d_c + h)}{(1 - e^2)(e^2 - 1) - 0.5(\sqrt{1 + 8Fr_1^2} - 1)} [33]$$

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Dividing Eq [33] by  $d_c$ , gives

$$\frac{d_1}{d_c} = \frac{-e^2(0.715d_c/d_c + h/d_c)}{(1 - e^2) - 0.5(\sqrt{1 + 8Fr_1^2} - 1)} [34]$$

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and finally substituting Eq [34] in Eq [1], yields

$$\frac{\Delta H}{H_{max}} = 1 - \frac{A + B}{\frac{3}{2} + \frac{Nh}{d_c}} [35]$$

329 where

$$A = \frac{-e^2(0.715 + h/d_c)}{(1 - e^2) - 0.5(\sqrt{1 + 8Fr_1^2} - 1)}$$

330

$$B = \frac{1}{2} \left( \frac{(1 - e^2) - 0.5(\sqrt{1 + 8Fr_1^2} - 1)}{e^2(0.715 + h/d_c)} \right)^2$$

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and N = the number of steps.

### 3. RESULTS AND DISCUSSION

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#### 3.1: Determination of the appropriate values of the Coefficients of Restitution (e) and the Froude No ( $Fr_1$ )

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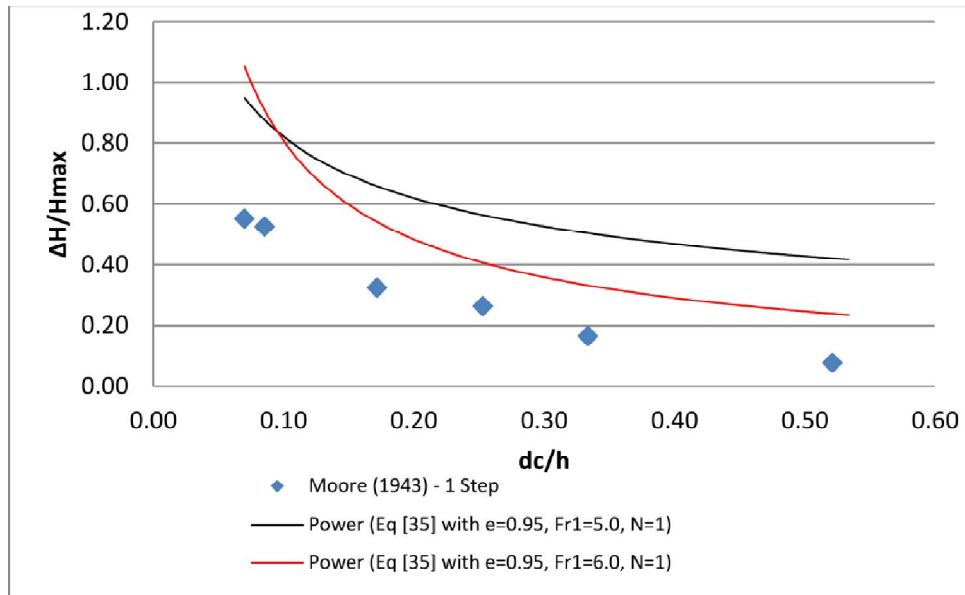
- a) The values of  $e = 0.95$  and  $Fr_1 = 6.0$  substituted in Eq [35] compared well with the measured data from Moore (1943) and were selected from Fig 3 to produce Fig. 4.
- b) The values of  $e = 0.70$  and  $Fr_1 = 16.0$  substituted in Eq [35] compared well with the measured data from Rand (1955) and were selected from Fig 5 to produce Fig. 6
- c) The values of  $e = 0.90$  and  $Fr_1 = 5.0$  substituted in Eq [35] compared well with the measured data from Stephenson (1979) and were from Fig 7. These values were then substituted in Eq [35] to produce Fig. 8.

Figures 3 to 9 showed that all the plotted curves followed the traditional concave shape distributions for rates of energy dissipation for all the flow rates (Chanson, 2001). They also showed that the rates of energy losses increased with an increasing number of steps and

350 decreased with increasing discharges, which agreed with the previous researchers (Matos,  
351 2000; Chanson, 2001b; Felder & Chanson, 2009a).

352

353 **Fig 3** depicted energy losses rates against flow rates plotted with the data from Moore  
354 (1943), the data sets from Eq [35] with the parameters of  $e = 0.95$ ,  $Fr_1 = 5.0$ ,  $N = 1$  as well as  
355 the data sets from Eq [35] with the parameters of  $e = 0.95$ ,  $Fr_1 = 6.0$ ,  $N = 1$ . As shown in the  
356 chart, the measured data sets and the data sets from Eq [35] with  $e = 0.95$ ,  $Fr_1 = 6.0$ , and  $N$   
357  $= 1$  were in close agreement. These parameters were, therefore, adopted for use in Eq [35]  
358 to produce the charts in Figures 2 and 8.  
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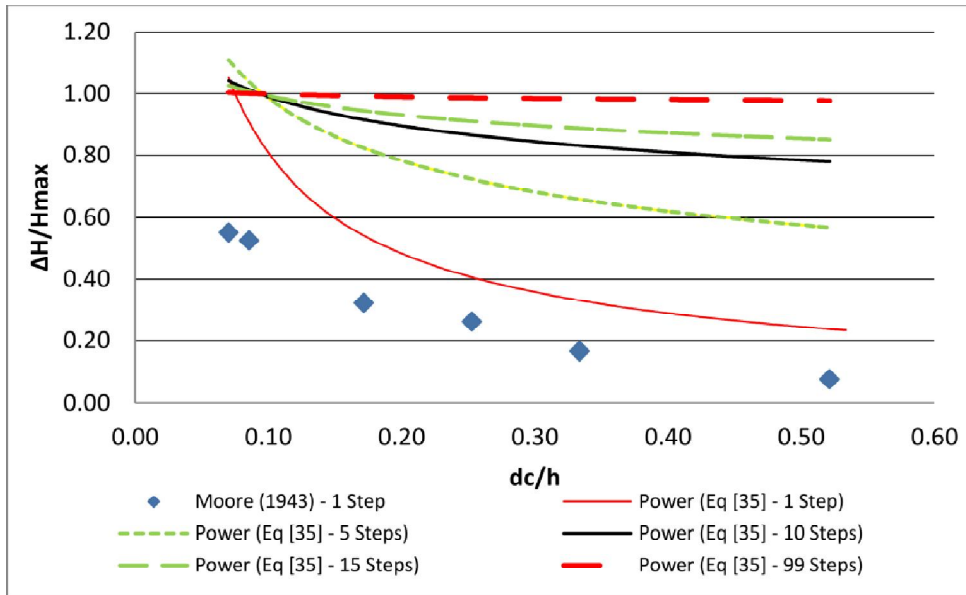
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362 **Fig.3. Variation of relative head loss,  $\Delta H/H_{max}$ , with  $d_c/h (= 0.07 - 0.52)$  for  $e = 0.95$ ,  $5.0$   
363  $< Fr_1 \leq 6.0$ ,  $N = 1$**

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365 **Fig 4** depicted energy losses rates against flow rates plotted with the data sets from Moore  
366 (1943) with  $N = 1$ , the data sets from Eq [35] with  $e = 0.95$ ,  $Fr_1 = 6.0$ , and  $N = 1, 5, 10, 15,$   
367 and 20.

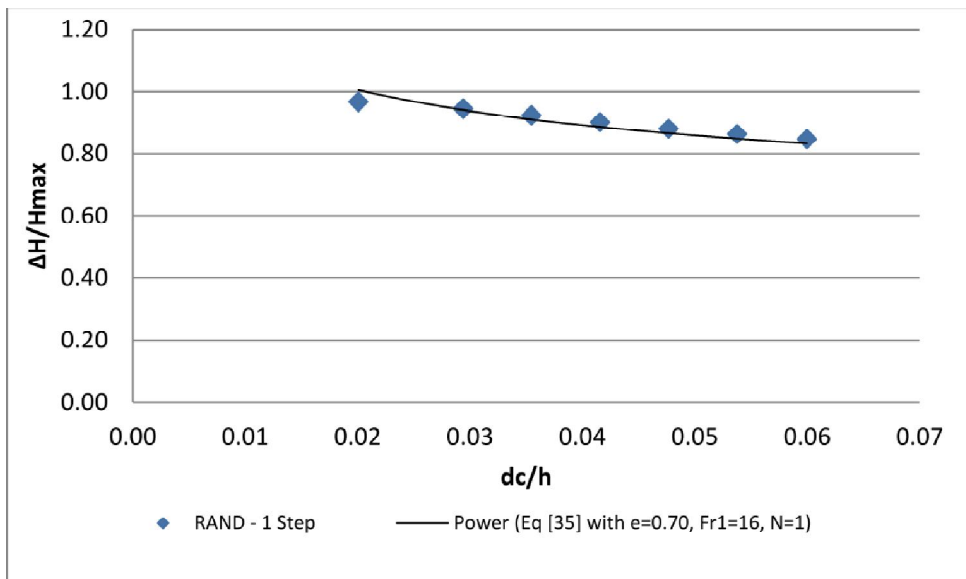
368 The new model grossly overestimated rates of energy losses for values of  $d_c/h$  less than  
369 0.20 and, therefore, should be used with caution for these values. The rates of these  
370 overestimation decrease with increasing flow rate,  $d_c/h$ . The model, however, predicted  
371 energy dissipation rates that compared fairly well with values of  $d_c/h$  between 0.20 and 0.53.  
372 The chart showed that about 100% energy dissipation rates were achieved with about  
373 number of steps,  $N = 99$ . Hence, 99 is the optimal number of steps needed to achieve about  
374 100% rates of energy dissipation with values of  $dc/h$  between 0.20 and 0.53.  
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**Fig.4. Variation of relative head loss  $\Delta H/H_{max}$  with  $d_c/h$  ( $= 0.07 - 0.52$ ) for  $e = 0.95$ ,  $Fr_1 = 6.0$ , and  $N$  between 1 and 15.**

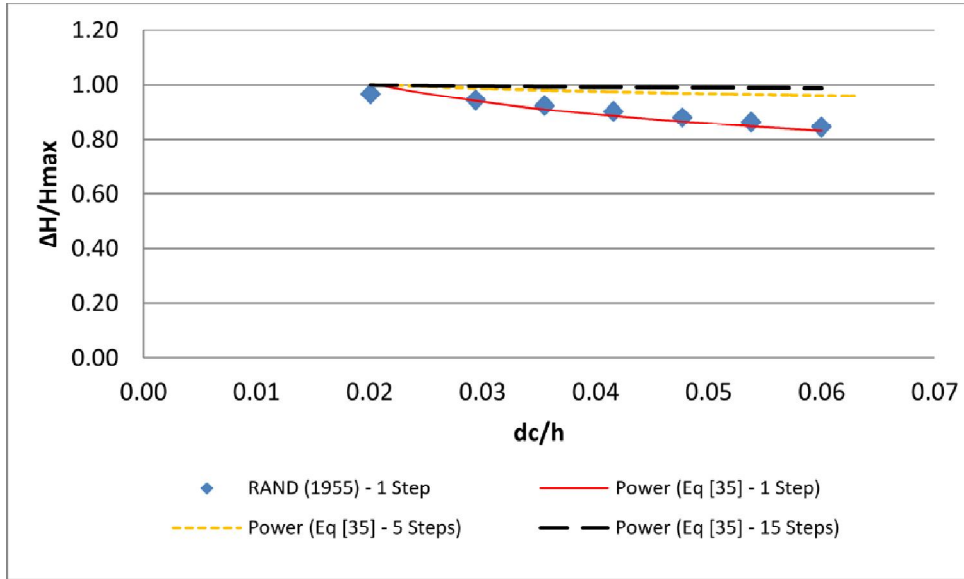
Fig 5 depicted energy loss rates against flow rates plotted with the data from Rand (1955) with  $N = 1$  and the data sets from Eq [35] with the parameters of  $e = 0.70$ ,  $Fr_1 = 16.0$ ,  $N = 1$ . As shown in the chart, the measured data sets and the data sets from Eq [35] with  $e = 0.70$ ,  $Fr_1 = 16.0$ , and  $N = 1$  were in good agreement. These parameters were, therefore, adopted for use in Eq [10] to produce the charts in Figures 5 and 6.



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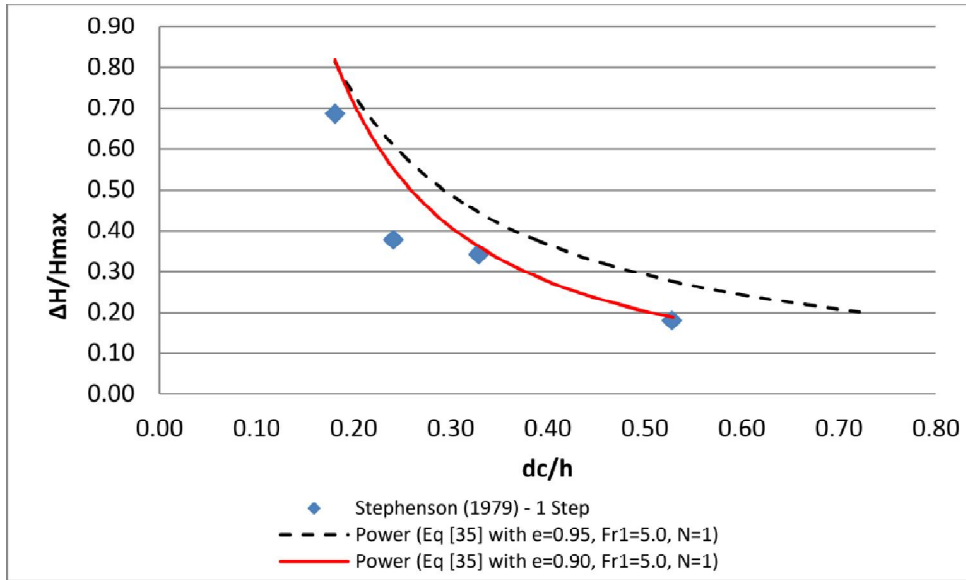
**Fig.5. Variation of relative head loss  $\Delta H/H_{max}$  with  $d_c/h$  ( $= 0.02 - 0.06$ ) for  $e = 0.70$ ,  $Fr_1 = 16.0$ ,  $N = 1$ .**

391 **Fig 6** depicted energy losses rates against flow rates plotted with the data sets from Rand  
 392 (1955) with  $N = 1$  and the data sets from Eq [10] with  $e = 0.70$ ,  $Fr_1 = 16.0$  and  $N = 1, 5, 10,$   
 393 and 15.  
 394 The new model predicted rates of energy dissipation predicted rates of energy dissipation  
 395 that compared well with the measured data for all values of  $d_c/h$  between 0.018 and 0.062.  
 396 The chart showed that about 100% energy dissipation rates were achieved with number of  
 397 steps,  $N = 15$ . Hence, 15 is the optimal number of steps needed to achieve about 100%  
 398 rates of energy dissipation with values of  $d_c/h$  between 0.018 and 0.062.  
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402  
 403 **Fig.6. Variation of relative head loss  $\Delta H/H_{max}$  with  $d_c/h$  ( $= 0.02 - 0.06$ ) for  $e = 0.70$ ,  $Fr_1$**   
 404  **$= 16.0$ ,  $N$  between 1 and 15**

405 **Fig 7** depicted energy losses rates against flow rates plotted with the data from Stephenson  
 406 (1979) with  $N = 1$ , the data sets from Eq [35] with  $e = 0.95$ ,  $Fr_1 = 5.0$ ,  $N = 1$  as well as the  
 407 data sets from Eq [35] with  $e = 0.90$ ,  $Fr_1 = 5.0$ ,  $N = 1$ . As shown in the chart, the measured  
 408 data sets and the data sets from Eq [10] with  $e = 0.90$ ,  $Fr_1 = 6.0$ , and  $N = 1$  were in close  
 409 agreement. These parameters were, therefore, adopted for use in Eq [35] to produce the  
 410 charts in Figures 7 and 8.  
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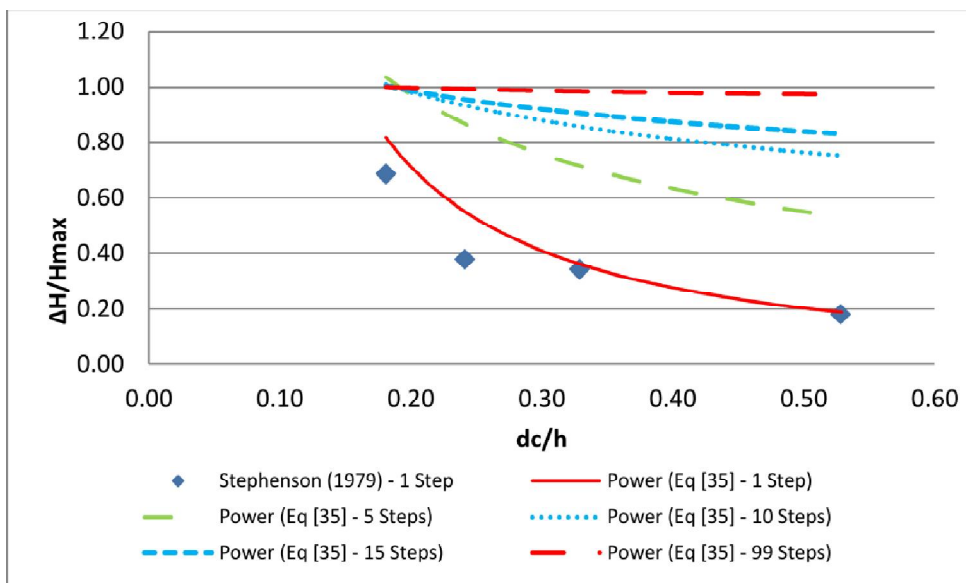
413 **Fig.7. Variation of relative head loss  $\Delta H/H_{max}$  with  $d_c/h (= 0.18 - 0.53)$  for  $0.90 < e \leq$**   
 414  **$0.90, Fr_1 = 5.0, N = 1$**

415 **Fig 8** depicted energy losses rates against flow rates plotted with the data sets from  
 416 Stephenson (1995) with  $N = 1$  and the data sets from Eq [35] with of  $e = 0.90, Fr_1 = 5.0$  and  
 417  $N = 1, 5, 10, 15,$  and  $20$ . The new model predicted rates of energy dissipation that agreed  
 418 with measured data for values of  $d_c/h$  of  $0.18$  and  $0.52$ .

419 The chart showed that about 100% energy dissipation rates were achieved with about  
 420 number of steps,  $N = 99$ . Hence,  $99$  is the optimal number of steps needed to achieve about  
 421 100% rates of energy dissipation with values of  $d_c/h$  between  $0.18$  and  $0.52$ .

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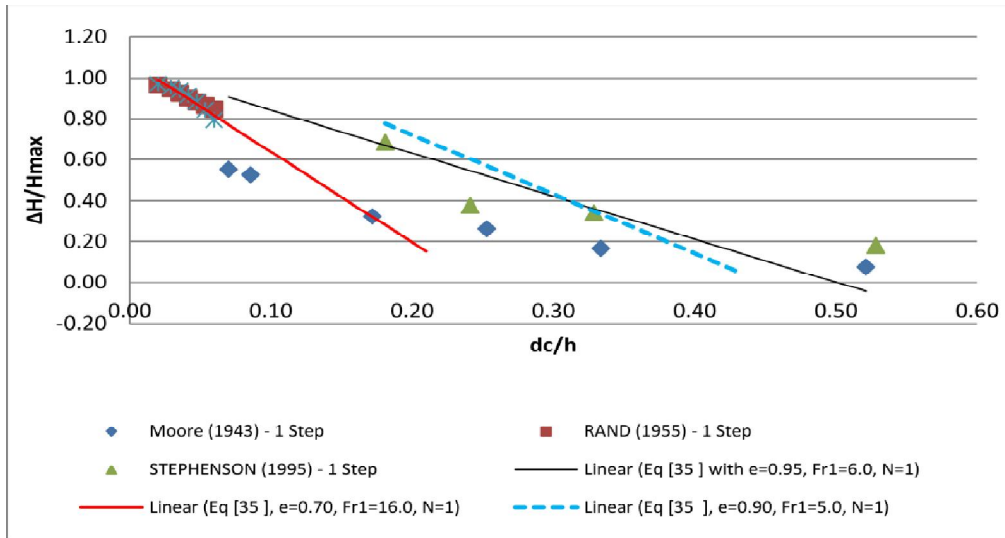
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425 **Fig.8. Variation of relative head loss  $\Delta H/H_{max}$  with  $d_c/h$  (= 0.18 – 0.52) for  $e = 0.90$ ,  $Fr_1$**   
 426 **= 5.0, and N between 15 & 99.**

427 **Fig 9** depicted energy losses rates against flow rates plotted with the data sets from Moore  
 428 (1943), Rand (1955), Stephenson (1979), and the data sets from Eq [35] with  $e = 0.95$ ,  $Fr_1 =$   
 429  $6.0$ ,  $N = 1$ ;  $e = 0.70$ ,  $Fr_1 = 16.0$ ,  $N = 1$ , and  $e = 0.90$ ,  $Fr_1 = 5.0$ ,  $N = 1$ . Data sets from the new  
 430 model compared well with measured data sets.  
 431 As indicated in the chart, measured data sets lay within the lower and upper boundaries of  
 432 the derived parameters. Model with parameters  $e = 0.95$ ,  $Fr_1 = 5.0$ ,  $N = 1$  and  $e = 0.90$ ,  $Fr_1 =$   
 433  $5.0$ ,  $N = 1$  bounded it at the top, while those with  $e = 0.70$ ,  $Fr_1 = 16.0$ , and  $N = 1$  bounded it  
 434 below.  
 435



436  
 437 **Fig. 9. Variation of relative head loss  $\Delta H/H_{max}$  with  $d_c/h$  (= 0.02 – 0.53) for  $e = 0.95$ ,  $Fr_1$**   
 438 **= 5.0, the measured data - MOORE (1943), RAND (1955), STEPHENSON (1995).**

- 440 It is, therefore, recommended as follows:
- 441 a) That these parameters,  $e = 0.95$  and  $Fr_1 = 5.0$ , be used in Eq [35] when  $d_c/h$  is  
 442 between 0.18 and 0.53 and
  - 443 b) That these parameters,  $e = 0.70$  and  $Fr_1 = 16.0$ , be used in Eq [35] when  $d_c/h$  is  
 444 between 0.02 and 0.06.

#### 445 4. CONCLUSION

446 A simple model, Eq [35], is developed for energy dissipation rates in a stepped spillway with  
 447 a developed hydraulic jump. The rates of energy losses increased with an increasing  
 448 number of steps and decreased with increasing discharges, which agreed with previous  
 449 researchers' works. New model data sets generally agreed with measured data sets: The  
 450 model predicted energy dissipation rates compared well; i) with values of  $e = 0.95$ ,  $Fr_1 = 6.0$ ,  
 451 and  $N$  between 1 and 99 for  $d_c/h$  between 0.07 and 0.53. The optimal number of steps  
 452 needed here for 100% energy dissipation is 99; ii) with values of  $e = 0.70$ ,  $Fr_1 = 16.0$  and  $N$   
 453 between 1 and 15 for  $d_c/h$  of 0.018 and 0.063. The optimal number of steps needed here for  
 454 100% energy dissipation is 15; and iii) with values of  $e = 0.90$ ,  $Fr_1 = 5.0$ , and  $N$  between 1  
 455 and 99 for  $d_c/h$  between 0.18 and 0.2. The optimal number of steps needed for 100% energy  
 456 dissipation is 99.  
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 459

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461

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464 the Department of Physics, The University of Sydney, Sydney, NSW 2006, Australia

465

466 **COMPETING INTERESTS**

467

468 The authors declare that there are no competing interests.

469

470 **AUTHORS' CONTRIBUTIONS**

471

472 Author 1 designed the study, performed the statistical analysis, wrote the protocol, and wrote  
473 the first draft of the manuscript. Author 2 managed the analyses of the study. All authors  
474 read and approved the final manuscript.

475

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 564

565 **LISTS OF SYMBOLS**

566

567 e – Coefficient of restitution (COR) assumed constant and less than 1 because of the elastic  
568 bounce of ball;

569  $d_c$  – critical height of ball (m);

570  $d_o$  - uniform ball depth at the weir (m);

571 Fr - Froude Number;

572 h - step height (m), which is also equals to the head loss at any intermediary step ;

573  $V_b$  - incident ball velocity ( $m/s^2$ );

574  $V_c$  - critical ball velocity ( $m/s^2$ )

575  $H_1$  – residual head at the bottom of the spillway (m);

576  $\Delta H$  – difference between the maximum head and the residual head (m);

577 H - total head (m);

578  $H_{max}$  - maximum head available (m):

579  $H_{max} = H_{dam} + 3/2 * d_c$ ;

580 Hres - residual head at the bottom of the spillway (m);

581 h - height of steps (m);

582 l - horizontal length of steps (m);

583 Q - discharge ( $m^3/s$ );

584 q - discharge per unit width ( $m^2/s$ );

585 Reynolds number defined as :  $Re = \rho_w * U_w * D_H / \mu_w$

586  $U_w$  - flow velocity (m/s):  $U_w = q_w/d$  ;

587 W - channel width (m);

588  $\Delta H$  - head loss (m);

589  $\mu$  - dynamic viscosity ( $N.s/m^2$ );

590  $\rho$  - density ( $kg/m^3$ );

591

592 Subscript

593 a – conditions at ball bounce;

594 b – conditions at step brink;

595 c – conditions at critical height;

596 d – conditions at maximum ball height;

597 N– number of step;

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